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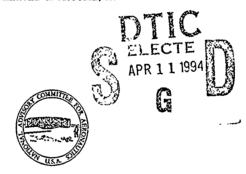
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NATIONAL ADVISORY COMMITTEE FOR AERONAUTICS

REPORT 1251

STRESS ANALYSIS OF CIRCULAR SEMIMONOCOQUE CYLINDERS WITH CUTOUTS

By HARVEY G. McCOMB, Jr.



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TN 3460) NACA Sept.

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Page 6, column 1, line 2: The beginning of the sentence starting on line 2 should be reworded to avoid the misinterpretation introduced by the word "same." The revised sentence should read as follows:

Because of symmetry, similar equations result when equation (1) is written for stringer j=1 at ring i=0 or for stringer j=0 at rings i=0 or i=1.

Page 23, column 1, line 3: Add the symbol j before the equal sign in the lower limit of the summation appearing in this equation.

STRESS ANALYSIS OF CIRCULAR SEMIMONOCOQUE CYLINDERS WITH CUTOUTS

By HARVEY G. McCOMB, Jr.

Langley Aeronautical Laboratory Langley Field, Va.

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National Advisory Committee for Aeronautics

Headquarters, 1512 II Street NW., Washington 25, D. C.

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REPORT 1251

STRESS ANALYSIS OF CIRCULAR SEMIMONOCOQUE CYLINDERS WITH CUTOUTS 1

By HARVEY G. McComb, Jr.

SUMMARY

A method is presented for analyzing the stresses about cutouts in circular semimonocoque cylinders with flexible rings. The method involves the use of so-called perturbation stress distributions which are superposed on the stress distribution that would exist in the structure with no cutout in such a way as to give the effects of a cutout. The method can be used for any loading case for which the structure without the cutout can be analyzed and is sufficiently versatile to account for stringer and shear reinforcement about the cutout.

INTRODUCTION

An airplane fuschage usually has openings or cutouts for entrance doors, cargo doors, windows, and many other purposes. The presence of such openings may result in a considerable redistribution of stress in the structure. Some knowledge of this stress redistribution is desirable in the structural design of fuschages near cutouts.

A large portion of the structure of many fusclages can be represented, approximately, by a circular semimonocoque cylinder, that is, a thin-walled circular cylinder stiffened by stringers (axial stiffening members) and rings (circumferential stiffening members). Some previous investigations relating to the problem of stress analysis of cylindrical semimonocoque shells with cutouts were reported in references 1-to-4. One limitation common to all of these analyses is that the flexibility of the rings or circumferential-stiffening members is neglected. In reference 5, Cicala discussed this limitation as well as certain other limitations in some of the previous investigations and introduced the idea that the effect of a cutout can be reproduced by superposing certain perturbation stress states on the stresses which would occur in the shell without a cutout.

The problem discussed by Cicala-in reference 5 is that of a cutout in a circular semimonocoque cylinder which is long in comparison to the length of the cutout. The analysis of reference 5 is somewhat limited because it can be used only for loading conditions which produce stringer stresses longitudinally antisymmetric about the center line of the cutout (for example, torsion), and it cannot take into consideration the effects of coaming stringer reinforcement. The present report is an extension of the approach of Cicala and presents

a method of analysis which can be used with more general loading conditions and with either shear or stringer reinforcement about the cutout.

In reference 6 the stress perturbation technique is applied to the analysis of stresses about cutouts in flat sheet-stringer panels under axial load. Three basic unit perturbation solutions were used as tools in this method of analysis. In part I of this report the analogous perturbation approach is described for the stress analysis of circular-semimonocoque cylinders with cutouts. The three perturbation-solution tools for circular semimonocoque cylinders analogous to those for the flat sheet-stringer panels of reference 6 are developed in part II of this report.

SYMBOLS

 \boldsymbol{A} effective cross-sectional area of a stringer 4* cross-sectional area of additional portion of a reinforced stringer $A_n=3B\delta^2-1+\cos n\delta$ $a_{ni} = \frac{\Delta_{ii} f_n(i)}{2}$ $(n \ge 2)$ $2L\sin\frac{n\delta}{9}$ $B_n=3B\delta^2+2(1-\cos n\delta)$ are distance-between stringers, Ro $(n \ge 2)$ $D_{rn} = \frac{1}{(rm+n)^2[(rm+n)^2-1]}$ coefficient in trigonometric series for by Ë Young's modulus of elasticity tangential force on ring i uniformly distributed between stringer j and stringer j+1 $f_n(i)$ coefficient in trigonometric series for stringer ionds shear modulus of elasticity

^{*}Supersedes NACA TN 3109, 1934 and NACA TN 7009, 1934 by Harvy, J. McComb, Jr., and NACA TN 3409, 1935 by Harvey G. McComb, Jr. and Emmet F. Low, Jr.

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II_1(n,\phi) = \sum_{n=-\infty}^{\infty} D_{rn} \cos{(rm+n)\phi}
H_2(n,\phi) = \sum_{n=0}^{\infty} (-1)^n D_{nn} \sin(n + n) \phi
              effective moment of inertia of a ring cross section
              longitudinal indices, indicating rings and bays
i,Ę
              has the value 1 when h is an integer and has the
Ĵ'n
                value 0 when h is not an integer
              circumferential indices, indicating stringers and
j,\eta
                panel rows
k,l,r,s
              integers
              distance between rings
M(i,\phi)
              bending moment in ring i
M_1, M_2
              applied moment and torque, respectively (see
                 fig. 5)
m
              total number of stringers in cylinder, m \ge 3
              index of terms in a trigonometric series
P
              external concentrated force in the longitudinal
                 direction applied to a stringer at its intersec-
                 tion with a ring, lb
              stringer load in stringer j at-ring i
p_{ij}
              basic stringer load in stringer i at ring i
\bar{p}_{ij}
              load in stringer j at ring i due to a unit-concen-
p_{ij}(\xi,\eta)
                 trated-perturbation load on stringer nat-ring &
              load in stringer j at ring i due to a wait shear
p_{ij}[\xi,\eta]
                perturbation load about shear panel (\xi,\eta)
              external shear force per unit length applied about
                 a shear panel, lb/in.
              shear flow in shear panel (i,j)
q_{ij}
              basic shear flow in shear panel (i,j)
\overline{q}_{ij}
              shear flow in shear panel (i,j) due to a unit
g_{ij}(\xi,\eta)
                 concentrated perturbation load on stringer n
                 at ring &
              shear flow in shear panel (i,i) due to a unit
q_{ij}(\xi,\eta)
                 shear perturbation load about shear panel
                 (\xi,\eta)
R
              radius to middle surface of sheet
S
              external force in the longitudinal direction
                 uniformly distributed along that portion of a
                 stringer which lies between adjacent rings, lb
S_n = \sum_{r=-\infty}^{\infty} D_{rn}^2
              thrust in ring i
T(\iota,\phi)
              thickness of sheet
              thickness of additional portion of a reinforced
                 shear panel, that is, a doubler plate
              thickness of all material carrying bending
ť
                 stresses in cylinder if uniformly distributed
                 around perimeter, A/b
              total stress energy
V(i,\phi)
              transverse shear in ring i
\alpha_{1n},\alpha_{2n},
              arbitrary constants
```

$$1-\frac{3}{2}\frac{B\delta^{2}}{\sin^{2}\frac{n\dot{\delta}}{2}}$$

$$\gamma_{n}=-2+\frac{12US_{n}}{12US_{n}}$$
Second central difference in the *i* direction or longitudinal direction, that is
$$\Delta_{i}g(i)=g(i+1)-2g(i)+g(i-1)$$

$$\delta \qquad \text{central angle between stringers, } 2\pi/m$$

$$\delta_{ii} \qquad \text{Kronecker delta; takes the value 1 when } r=s$$
and takes the value 0 when $r\neq s$

$$\zeta_{n}=\pm e^{-\psi_{n}}$$

$$\Lambda_{1n},\Lambda_{2n} \qquad \text{quantities defined immediately following equation } (24)$$

$$\phi \qquad \text{angular coordinate for rings}$$

$$\chi_{n}=\frac{1}{2}\cos^{-1}\left[\frac{\beta_{n}-1}{2}-\sqrt{\left(\frac{\beta_{n}+1}{2}\right)^{2}-\gamma_{n}^{2}}\right] \qquad (D_{n}>1)$$

$$=\frac{1}{2}\cosh^{-1}\left[\frac{\beta_{n}-1}{2}-\sqrt{\left(\frac{\beta_{n}+1}{2}\right)^{2}-\gamma_{n}^{2}}\right] \qquad (D_{n}<1)$$

$$\psi_{n}=\frac{1}{2}\cosh^{-1}\left[\frac{\beta_{n}-1}{2}+\sqrt{\left(\frac{\beta_{n}+1}{2}\right)^{2}-\gamma_{n}^{2}}\right]$$

BASIC ASSUMPTIONS

A structure of the type considered in this report is shown in figure 1. It consists of a thin-walled circular cylinder stiffened by stringers in the longitudinal direction and by rings in the circumferential direction. The rings and stringers divide the thin-walled shell into rectangular panels which are called shear panels. The cutout is assumed to be rectangular—it removes an arbitrary number of shear panels and interrupts the corresponding stringers.

Some loading conditions which can be handled with this method of analysis are illustrated in figure 1. Other loading conditions are permissible if the stress distribution in the cylinder without the cutout is known.

A typical portion of the structure is shown in-figure 2 with the index system used in this report to designate stringers, rings, bays, and panel rows. Note that the intersection of

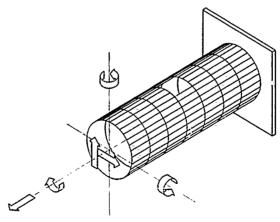


FIGURE-1.-Circular semimonocoque cylinder with cutout.

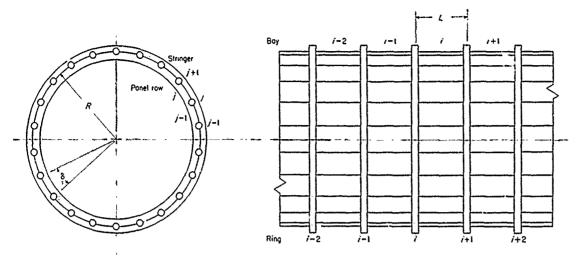


FIGURE 2 .- Portion of typical cylinder.

ring i and stringer j occurs at the lower left-hand corner of shear panel (i,j).

The analysis is based on the following assumptions regarding the properties of the structure:

- (a) The cylinder is long relative to the length of the cutout.
- (b) The stringers are uniform and equally spaced around the shell, and the sheet is of constant thickness.
- (e) The stringers carry only direct stress, and the sheet takes only shear stress which is constant within each shear panel; thus stringer stresses vary linearly between adjacent rings.
- (d) The rings are uniform and have a finite bending stiffness in their own planes, but they do not restrain longitudinal displacements of the stringers. The bending of the rings is inextensional.
- (e) The difference between the radius to the middle surface of the sheet and the radius to the neutral axis of a ring is negligible.
 - (f) The structure is elastic and no buckling occurs.

I—ANALYSIS OF STRESSES ABOUT CUTOUTS BY A PERTURBATION LOAD TECHNIQUE

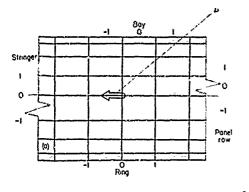
PERTURBATION STRESS DISTRIBUTIONS

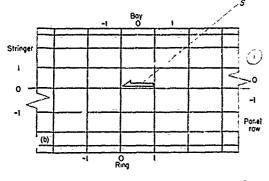
The tools for the method of analysis to be described are the stress distributions due to three types of loads, called perturbation loads, applied to an infinitely long circular cylinder with no cutout. One perturbation load consists of a concentrated force P imposed on one stringer of the shell at its intersection with a ring, the force acting in the direction of the stringer. This load is illustrated in figure 3 (a) and is called the concentrated perturbation load. A second type illustrated in figure 3 (b), is called the distributed perturbation load and consists of a force S uniformly distributed along the portion of one stringer which extends between two adjacent rings, the force acting in the direction of the stringer. The third type, shown in figure 3 (c), is called the shear

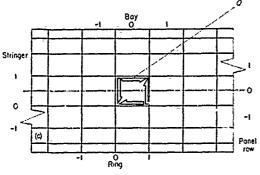
perturbation load and consists of uniformly distributed forces per unit length Q applied along the stringers and rings that border one shear panel of the shell, the forces acting in such a way as to cause pure shear in that panel.

For each of the three perturbation loads, formulas are developed in part 11-of this report-which give stringer-loads in every stringer at each ring and shear flows in each shear panel of the shell. By use of these formulas, tables of coefficients can be computed which give stringer loads and shear flows in the neighborhood of each perturbation load due to a unit magnitude of that load. Such tables for a cylinder having 36 stringers and various values of the structural parameters B and C are presented as tables 1 to 30. These tables were calculated on an IBM Card-Programmed Electronic Calculator. The application of these tables is not limited to cylinders with 36 stringers. In general, the total stringer area can simply be redistributed into 36 fictitious stringers. The values of the parameters B and C are not changed by such a redistribution of stringer area. Then the tables can be thought of as presenting (a) the load which is taken by all of the normal-stress-carrying material up to 5° on either side of the location of a fictitious stringer and (b) the shear flows at points in the sheet halfway between fictitious stringers.

Part (a) of each-table contains the values of p_{ij} and $q_{il}L$ due to a concentrated perturbation load P=1 on stringer j=0 at ring station-i=0. Part (b) contains the values-of p_{ij} and $q_{il}L$ due to a distributed perturbation load of total magnitude S=1 on stringer j=0 between rings i=0 and i=1. Part (c) contains the values of p_{ij}/L and q_{ij} due to a shear perturbation load per unit length of magnitude Q=1 about shear panel (0,0). The positive-senses of the perturbation loads are the senses shown in figure 3; stringer loads are assumed positive in tension, and shear flow is positive when an element of the sheet is loaded by shears which act







(a) Concentrated, (b) Distributed, (c) Shear,

Frounc 3 .- Perturbation loads.

in the positive sense of the shear perturbation load. The solutions for arbitrary locations of the perturbation loads are readily obtained from the tables by means of changes of indices.

The application of these-perturbation loads and the stress distributions caused by them in the stress analysis of eircular semimonocoque cylinders with cutouts is discussed in the following section. The perturbation solutions are exact only for infinitely long cylinders. However, in the solution of a cutout problem, the perturbation loads are applied in self-equilibrating groups in order not to disturb the overall equilibrium of the structure; therefore, the stresses due to

the perturbation loads decay-rapidly in the longitudinal direction. Consequently, the application of perturbation stress distributions for an infinitely long cylinder to a cylinder of finite length is justified if the vicinity of application of the perturbation loads is far from the ends of the cylinder.

METHOD OF ANALYSIS

STRUCTURE WITH NO REINFORCEMENT ABOUT CUIOUT

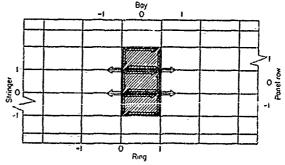
Application of perturbation loads.--Consider, first, a structure like that shown in figure 1 which has no reinforcement about the cutout. The stress distribution in such a shell can be thought of as a superposition of the stresses which would exist in the structure without a cutout and perturbation stress distributions which arise because of the outout. The structure without a cutout is called herein the basic structure. The stress-distribution which would existin this structure is called herein the basic stress distribution. In the present report the basic stress distribution is assumed to-be known. Then the problem of analyzing a structure with a cutout consists of the determination of the perturbation stress distributions to be superposed on the basic stresses in such a manner as to annihilate the effects of that portion of the basic structure which-lies within the boundaries of the catout. Finding the proper magnitudes of these perturbetion stresses involves the solution of a system of simultaneous algebraic equations.

At the cutout boundary in the structure with the cutout, two conditions must be satisfied: (a) the stringer load must be zero at points where a stringer is interrupted by the cutout and (b) no external shear forces may act on portions of stringers and rings which border the cutout. By superposing concentrated and shear perturbation loads on the basic structure, the resultant stresses can be made to satisfy these conditions.

The method of analysis is as follows:

(1) Find the stress distribution for the basic structure, that is, the cylinder without a cutout.

(2) Place perturbation loads on the basic structure in the following manner: At each point where a stringer would be interrupted by the cutout, place a concentrated perturbation load; and, about each shear panel which would be removed by the cutout, place a shear perturbation load. For the case of a cutout removing three shear panels and interrupting two stringers, these perturbation loads are shown in figure 4.



Floure 4 .- Application of perturbation loads.

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(3) With the use of the tables of coefficients, write a set of simultaneous algebraic equations which state the following conditions:

(a) At the points where a stringer is to be interrupted by the cutout boundary, the resultant stringer load must vanish when the boundary is approached from the structure outside of the cutout. This resultant stringer load is composed of the basic stringer load plus the stringer-load due to all the perturbation loads.

(b) In each shear panel which is to be removed by the cutout, the basic shear flow plus the shear flow due to all the perturbation loads must be equal to the shear perturbation load applied to the portions of stringers and rings which border that given panel. Thus, the shear flow exerted by the shear panel on the portions of stringers and rings bordering it will exactly cancel the shear perturbation load applied to those same portions of stringers and rings.

(4) Solve the system of equations from step (3) for the magnitudes of the perturbation loads, and superpose the stress distributions due to these loads on the basic distribution. This procedure yields the stress distribution in the structure with cutout.

Upon-completion of these four steps, the magnitudes of the perturbation loads on the brsic structure bave been adjusted so that simultaneous-removal of that portion of the basic structure which lies within the cutout boundary and the perturbation loads themselves would not disturb the remainder of the structure. The perturbation loads are in equilibrium-with-the portion-of the basic structure lying-within the cutout-boundary. The stresses outside the cutout boundary in the basic structure subjected to the actual external loading together with the perturbation loads are precisely the same as the stresses in the structure with the cutout subjected to the external loading alone.

Conditions 3 (a) and 3 (b) can be expressed mathematically by the following equations, respectively:

$$\sum_{\xi} \sum_{\eta} P_{\xi\eta} p_{\xi j}(\xi, \eta) + \sum_{\xi} \sum_{\eta} Q_{\xi\eta} p_{\xi j}[\xi, \eta] + \overline{p}_{\xi j} = 0$$
 (1)

$$\sum_{\xi} \sum_{\eta} P_{\xi,\eta} q_{ij}(\xi,\eta) + \sum_{\xi} \sum_{\eta} Q_{\xi,\eta} q_{ij}[\xi,\eta] + \overline{q}_{ij} = Q_{ij}$$
 (2)

The unknowns are $P_{t\eta}$, the magnitude of the concentrated perturbation load on stringer η at ring ξ , and $Q_{t\eta}$, the magnitude of the shear perturbation load about shear-panel (ξ,η) . The coefficients $p_{i\theta}(\xi,\eta)$ and $q_{i}(\xi,\eta)$ are found in part (a) of the tables and the coefficients $p_{i\theta}(\xi,\eta)$ and $q_{i\theta}(\xi,\eta)$ are found in part (c). The summations in each case are extended over the appropriate perturbation loads. Equation (1) is written for each i,j where a stringer is to be interrupted by the cutout and refers in each case to the stringer load as the point i,j is approached from within that portion of the structure lying outside-the cutout boundary. Equation (2) is written for each i,j where a shear panel is to be removed by the cutout. The form of equations (1) and (2) is the same regardless of whether the rings in the cylinder are considered rigid or flexible.

This method of analysis may be applied to a cylinder having a cutout more than 1 bay long, but, in such a situation, the effects of removing ring segments from the region within

the cutout boundary are reglected. In the rigid-ring case, such effects do not exist if the cut rings remain effectively rigid: in the flexible-ring case, the effects of cutting a ring could in principle, be taken into account through the introduction of additional types of perturbation loads. It is possible that even with flexible rings the effects of cutting a ring are negligible in certain cases, but this would have to be verified by further investigation.

Sample calculation.—In order to illustrate the method of calculation, the cylinder shown in figure 5 is analyzed. A cutout which removes three shear panels and interrupts two stringers is located in the central bay. The properties of the cylinder are taken as follows:

$$m=36$$
 $A=0.260$ sq in.
 $R=15$ in.
 $L=12$ in.
 $t=0.051$ in.
 $b=R\frac{2\pi}{36}=2.62$ in.
 $t'=\frac{0.260}{2.62}=0.0992$

For the purposes of this example suppose the rings are very heavy and can be considered rigid in bending in their own planes. From these properties the structural parameters B and C are calculated. The table corresponding to the values of B and C closest to the computed values will be used. If E is taken as 10.6×10^6 psi and C is taken as 4×10^6 psi, the parameters B and C are

$$B = \left(\frac{10.6}{4}\right) \left(\frac{0.0902}{0.061}\right) \left(\frac{15}{12}\right)^2 = 8.05$$

C=

Suppose that the cylinder is loaded with the bending moment M_1 and torque M_2 shown in figure 5. The perturbation load system for this problem is shown in figure 4. The concentrated perturbation loads are doubly symmetric about the cutout. The shear perturbation loads are symmetric about panel row j=0. Let P represent the magnitude of each of the concentrated perturbation loads. Let Q_0 represent the magnitude of the shear perturbation load about shear panel (0,0); and let Q_1 represent the magnitude of the shear perturbation loads about shear panels (0,1) and (0,-1).

Equations (1) and (2) are now written for this example by use of the tables of coefficients for B=8 and C=0. Equation (1) for the stringer load condition in stringer j=1 at ring i=1 is written with the aid of tables 1 (a) and 1 (c) as follows:

$$-0.5000P + 0.0476P + 0.0895P + 0.1192Q_1L - 0.1192Q_0L - 0.0374Q_1L + \overline{p}_{11} = 0$$

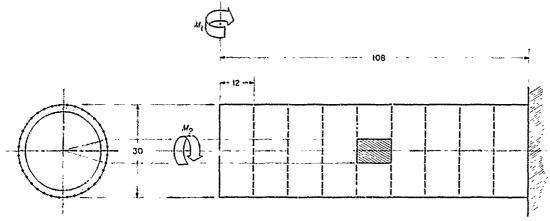


FIGURE 5.- Circular cylinder with cutout used in sample calculation.

where \overline{p}_{11} is the basic stringer load in stringer j=1 at station i=1. Because of symmetry the same equation, results when equation (1) is written for stringer j=1 at ring i=0 or for stringer j=0 at rings i=0 or i=1. Equation (2) for shear panel (0,0) is

$$-0.2262 \frac{P}{L} + 0.2262 \frac{P}{L} - 0.2262 \frac{P}{L} + 0.2262 \frac{P}{L} + 0.6986 Q_0 - 2(0.0629) Q_1 + \ddot{q}_{\infty} = Q_0$$

where \bar{q}_{00} is the basic shear-flow in shear-panel (0,0). For shear panels (0,1) and (0,-1), equation (2) gives

$$-0.2262 \frac{P}{L} + 0.2262 \frac{P}{L} - 0.1368 \frac{P}{L} + 0.1368 \frac{P}{L} + 0.6986 Q_1 -$$

 $-0.0629Q_0 + 0.0119Q_1 + \overline{q}_{01} = Q_1$

where \bar{q}_{01} is the basic shear flow in shear panel (0,1). These three equations in the three unknowns P, Q_0 , and Q_1 become

$$\begin{array}{c}
0.3629P + 0.1192Q_0L - 0.0818Q_1L = \overline{p}_{11} \\
0.3014Q_0L + 0.1258Q_1L = \overline{q}_{00}L \\
0.0629Q_0L + 0.2895Q_1L = \overline{q}_{01}L
\end{array}$$
(3)

For simplicity, let $M_1 = M_2 = 100,000$ lb-in. In the present example, the basic stress distribution can be found from elementary beam and torsion theories which give $\bar{p}_{11} = 370$ pounds and $\bar{q}_{00} = \bar{q}_{01} = 70.3$ lb/in. When these constants are introduced into the system of equations (3), the solution is

$$P=1,020 \text{ lb}$$
 $Q_6L=1,750 \text{ lb}$
 $Q_1L=2,560 \text{ lb}$

Stringer loads and shear flows in the neighborhood of the cutout are obtained by superposing the effects of these perturbation loads on the basic stress distribution. For example, with the use of tables 1 (a) and 1 (c) the stringer load at the intersection of ring i=0 and stringer j=2 is

given by

$$P(0.0895 + 0.0511) + Q_1L(0.1192 + 0.0125) + Q_0L(0.0374) + \overline{p}_{02}$$

= $545 + \overline{p}_{02}$

The basic stringer load \bar{p}_{cc} equals 358 pounds. Therefore, the load in stringer j=2 at ring i=0 is 903 pounds. Other stringer load, at ring i=0 are shown in figure G(a). The shear flow in shear panel (-1,1) is given by

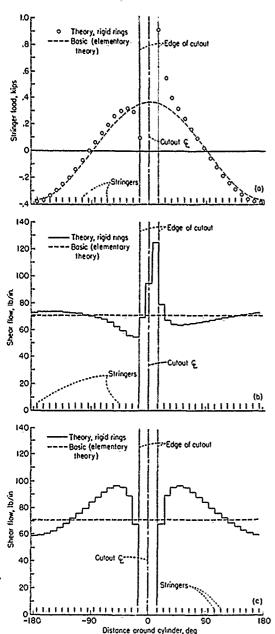
$$\frac{1}{L} [P(0.2262+0.1368 \div 0.0044-0.0360) + Q_1L(0.1357-0.0159) + Q_0L(0.0097)] + \overline{q}_{-1,1} = 55.1 + \overline{\zeta}_{-1,1}$$

The basic shear flow $\bar{q}_{-1,1}$ equals 70.8 lb/in. Thus, the shear flow in panel (-1,1) is 125.9 lb/in. Other shear flows in bay i=-1 are shown in figure 6 (b), and in figure 6 (c) are presented shear flows in the net section-(bay i=0).

STRUCTURE WITH REINFORCEMENT ABOUT CUTOUT

Shear reinforcement.—The method of analysis is easily extended to problems where shear panels are reinforced in the neighborhood of the cutout. Suppose that some of the shear panels around the cutout are reinforced by the addition of a certain thickness of sheet (i. e., a doubler plate). Then, the procedure consists of adding shear perturbation loads to-each of these shear panels in the basic structure. On the doubler plates is placed the same shear perturbation load except with opposite sign. Then, for each reinforced shear panel, an equation is written which states the requirement that the shear stress in the shear panel of the basic structure shall count the shear stress in the doubler plate used to reinforce that panel. When this condition is satisfied, the loaded doubler plates can conceptually be inserted into the basic structure without disturbing continuity. The shear perturbation loads on the doubler plates cancel the shear perturbation loads on the basic structure.

As an example, consider for simplicity the cylinder shown in figure 5 loaded only with bending moment M_1 . The most highly-loaded shear panels are those indicated by the vertical



(a) Stringer loads at ring bordering cutout (ring i=0).
(b) Shear flow in bay adjacent to cutout (bay i=-1).
(c) Shear flow in net section (bay i=0).

FIGURE 6.—Results of sample calculation.

hatching-in-figure 7. Suppose, now, that these shear panels are reinforced by the addition-of plates of thickness t^* to the skin of thickness t so that the total thickness in these shear panels is $t+t^*$. The perturbation load system to be placed on the basic structure is shown in-figure 8. The four $\frac{359282-56-2}{2}$

doubler plates of thickness t* are shown as free bodies in figure 8. The shear-perturbation loads applied to them are of the same magnitude as those applied to the basic portions of the reinforced shear panels, but are opposite in sign. The conditions that must be satisfied are:

(a) The stringer load is zero in stringers j=0 and j=1 at rings i=0 and i=1 as each of these points is approached from the structure outside of the cutout.

(b) The shear flow in shear panels (0,-1), (0,0), and (0,1) cancels any shear perturbation load applied about these panels (In this example, no shear is developed in the shear panels of bny i=0 and this condition is automatically satisfied.)

(c) The shear stress in each of the shear panels (i,1), (1,-1), (-1,1), and (-1,-1) in the basic structure must equal the shear stress in the corresponding doubler plate.

Condition (a), which must hold where stringers j=0 and j=1 are interrupted by the cutout, is expressed by a single equation because of symmetry:

$$(-0.5000+0.0476+0.0895)P+(-0.1192-0.0374+0.0067-0.0118)QL+\vec{p}_{11}=0$$

where P and Q are the magnitudes of the concentrated and shear perturbation loads, respectively, and $\overline{P}_{\rm H}$ is the basic stringer load. The condition-in-shear panel (1,1) that the shear stress in the basic portion of the sheet equals the shear stress in the doubler plate (condition (c)) is expressed as

$$\left[\langle -0.2262 - 0.1368 - 0.0044 + 0.0360 \rangle \frac{P}{L} + \\ (0.6986 - 0.0119 - 0.0068 + 0.0052)Q \right] \frac{1}{l} = -Q \frac{1}{l^*}$$

where t is the thickness of the basic portion of the shear panel and t^* is the thickness of the doubler plate. Because of symmetry, the same equation expresses condition (c) for the other three reinforced shear panels. These equations become

0.3629
$$P$$
+0.1617 $QL = \overline{p}_{\text{II}}$
-0.3314 P + $\left(\frac{t}{t^*}$ +6.6851 $\right)$ QL =0

For a given value of t/t^* and for a given magnitude of M_1 (so that \overline{p}_H can be computed), this system of equations can be solved for P and Q, and the stress distributions due to these perturbation loads can then be superposed on the basic stress distribution to give the stresses about the cutout.

Stringer reinforcement.—The method of analysis is also easily extended to problems where stringers are reinforced in the neighborhood of the cutout. For example, suppose the coaming stringers in the structure shown in figure 5 have reinforcement of constant cross-sectional area extending 1 bay on either side of the cutout. This coaming-stringer reinforcement is illustrated in figure 9. Let the area of the added reinforcing portion of a coaming stringer be A^* so that the total area of the reinforced portion of the stringer is $A+A^*$. It is assumed that the stringer load is abruptly transmitted into the added portion of the reinforced coaming stringer so that the stress is always given by the force divided by the cross-sectional area.

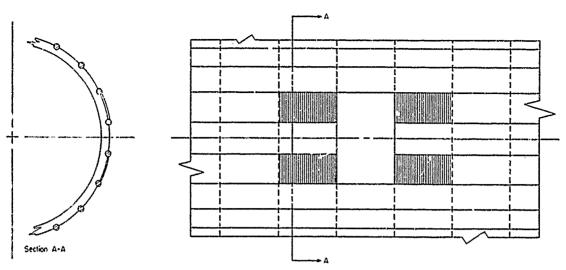


FIGURE 7.-Cutout with shear reinforcement.

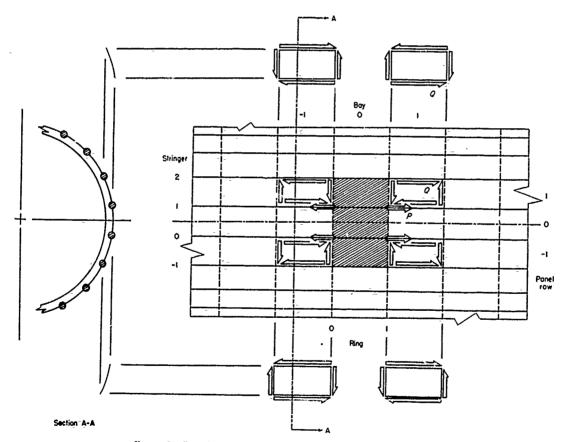


Figure 8.—Perturbation load system for a problem of shear reinforcement.

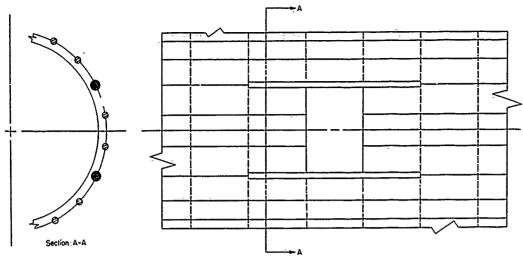


FIGURE 9.-Cutout with reinforced coaming stringers.

Again for simplicity suppose that the cylinder is loaded only by the bending moment M_1 shown in figure 5. The perturbation load system to be placed on the basic structure is shown in figure 10. The added reinforcing portions of the coaming stringers are shown as free bodies in figure 10 with the proper perturbation loads applied to them The conditions that must be satisfied are:

- (a) The stringer load is zero in stringers j=0 and j=1 at rings i=0 and i=1 as each of these points is approached from the structure outside of the cutout.
- (b) The shear flow in shear panels (0,-1), (0,0), and (0,1) cancels any shear perturbation load applied about these shear panels. (This condition is automatically satisfied in this example.)
- (c) The stress in the basic portions of the coaming stringers j=-1 and j=2 equals the stress in the added reinforcing portions at rings: i=0 and i=1.
- (d) In the basic portions of the coaming stringers j=-1 and j=2 at rings i=-1 and i=2, when these points are approached from the side which is reinforced, the stress equals the stress at the ends of the added reinforcing portions of the coaming stringers.

Because of the symmetry in this structure, only three equations are required. The unknowns are P_1 and P_2 , the magnitudes of the concentrated perturbation loads, and S, the magnitude of the distributed perturbation loads. Condition (a), which must hold where stringer j=1 is interrupted by the cutout, is expressed with the use of tables 1(a) and 1(b) as follows:

$$(-0.5000 + 0.0476 + 0.0895)P_1 + (-0.0895 - 0.0511 - 0.0490 - 0.0475)P_2 + (-0.0727 - 0.0340 - 0.0629 - 0.0499)S + \overline{p}_{11} = 0$$

The condition that the stringer stress in the basic portion of stringer j=2 equals the stress in the added reinforcing portion at ring i=1 (condition (c)) is expressed as

$$\begin{array}{l} [(0.0895 + 0.0511)P_1 + (-0.0476 - 0.0330 - 0.0565 - 0.0402)P_2 \\ + (-0.1924 - 0.0195 - 0.0567 - 0.0379)S + \tilde{p}_{13}] \frac{1}{A} = (P_2 + S) \frac{1}{A^*} \end{array}$$

Finally, the condition that the stress in the basic portion of stringer j=2, as the ring i=2 is approached from the reinforced side, equals the stress at the ends of the added reinforcing member (condition (d)) is expressed as follows:

$$\begin{array}{l} [(-0.5000 - 0.0459 - 0.0394) \, P_t + (0.1924 + 0.0195 - 0.0499 \\ -0.0398) S + (-0.0895 - 0.0511 + 0.0490 + 0.0475) P_1 + \overline{p}_{22}] \, \frac{1}{A^2} = \frac{P_2}{A^2} \end{array}$$

These three equations become

$$0.3629P_1 + 0.2371P_2 + 0.2195S = \overline{p}_{11}$$

$$-0.1406P_1 + \left(\frac{A}{A^*} + 0.1773\right)P_2 + \left(\frac{A}{A^*} + 0.3065\right)S = \overline{p}_{12}$$

$$0.0441P_1 + \left(\frac{A}{A^*} + 0.5853\right)P_2 - 0.1222S = \overline{p}_{22}$$

When A/A^* is known and the magnitude of the external moment M_1 is known so that the basic stringer loads \overline{p}_{11} , \overline{p}_{12} , and \overline{p}_{22} can be computed, this system of equations can be solved for the unknowns P_1 , P_2 , and S. Superposition of the stresses due to these perturbation loads on the basic stress distribution yields the stresses about the cutout.

In this example the basic stringer loads do not vary in the longitudinal direction, and the concentrated and distributed perturbation loads can be applied in pairs, equal in magnitude and opposite in sign, as shown in figure 10. However, in cases where the basic stringer loads do vary longitudinally, for example, when the shell is loaded in shear and bending, the concentrated and distributed perturbation loads may not occur in equal and opposite pairs. Furthermore, additional distributed perturbation loads may be necessary on the coaming stringers in bay i=0. If such is the case, the stress conditions which were used in the example no longer-provide a sufficient number of equations to determine the magnitudes of the perturbation loads. The required

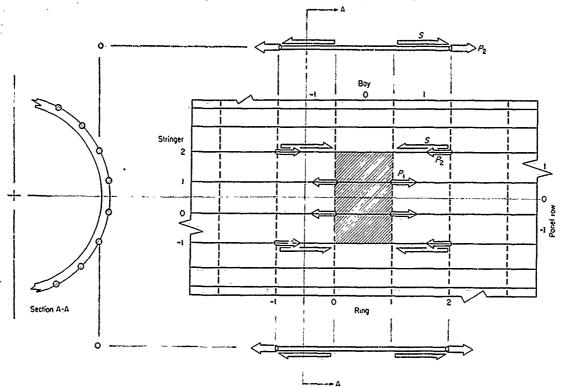


FIGURE 10.—Perturbation load system for a problem of coaming-stringer reinforcement.

supplementary equations are found from the conditions of equilibrium obtained when the added reinforcing portions of the coaming stringers are considered as free bodies.

comparison of results for reinforced and unreinforced structures.—Some calculated results for the problems of cutouts with reinforcement just discussed are compared with the results for the structure without reinforcement in the following tables;

	St	ringer load, lb, fo	r	
Intersection of ring and stringer	Structure without rein- forcement	Structure with reinforced omming stringers,	Structure with shear reinforcement,	
(1,2) (1,3) (1,4) (1,5) (1,6)	501 122 359 303 211	758 331 296 238 209	\$07 422 359 302 212	

	Sh	ear flow, lb/in., f	ж—
Siwar panel	Structure without rein- forcement	Structure with reinforced coaming stringers,	Structure with shear rein- forcement, t'=t
(1, 0) (1, 1) (1, 2) (1, 3) (1, 4)	-04.1 -12.3 -5.6 -2.5	-27.3 3 .4 .5	0 -30 6 -13.3 -5.8 -2.3

The reinforced shear panels were assumed to have sheet twice as-thick as the uniform sheet, the-reinforced portions of the coaming stringers were taken to have twice the area of the uniform stringers. The applied-bending moment M_1 was taken as 100,000 lb-in.

The following comparison is noted for these illustrative examples. In the case of coaming-stringer reinforcement, the maximum stringer load is increased, but the maximum stringer stress is decreased (because stringer area is doubled), and the maximum shear flow is not appreciably changed. In the case of shear reinforcement, the maximum shear flow is increased only slightly so that maximum shear stress is considerably reduced, and stringer loads are not appreciably affected.

II—DERIVATION OF PERTURBATION SOLUTIONS ANALYTICAL APPROACH

Equations for the stress distributions arising from the three perturbation loads illustrated in figure 3 are derived in this part of the report. The perturbation solutions are obtained by use of the principle of minimum complementary energy. This principle states that, among all possible stress distributions in the structure which satisfy equilibrium and the boundary conditions on stress, the distribution that most nearly satisfies compatibility is the one which minimizes the complementary energy π^* where

$$\pi^*$$
=Internal energy-(Work done by surface stresses) (4) (acting through the prescribed surface displacements

Since displacements are not prescribed anywhere on the structure, the second term on the right-hand side of equation (4) is omitted. The complementary energy becomes the internal energy or stress energy of the structure.

In writing the equation for the stress energy, the following factors are considered: the energy of axial distortion of the stringers, the shear energy in the sheet, and the bending energy of the rings in their own planes. Each of the perturbation loads is shown in its positive sense in figure 3. Stringer loads are taken as positive in tension. Shear flows are positive as shown in figure 11. Ring bending moments, shear, and thrusts are placed on the ring element in figure 11 in the positive sense. The stress energy in the structure can be expressed as

$$U = \sum_{1 = -\infty}^{\infty} \sum_{j=0}^{m-1} \left[\frac{L}{6AE} \left(p_{ij}^2 + p_{ij} p_{i+1,j} + p_{i+1,j}^2 \right) + \frac{R\delta L}{2Gt} q_{ij}^2 \right] +$$

$$\sum_{j=-\infty}^{\infty} \int_{0}^{2\pi} \frac{R}{2EI} M^2(i,\phi) d\phi$$
(5)

where the integration over the length of a stringer between adjacent rings has been carried out.

In the analysis to follow, stringer loads are expressed in the form of a finite trigonometric series. Then, by using the equations of statics, the shear flows and ring bending moments are written in terms of the coefficients of this trigonometric series. The expression for stress energy, equation (5), is minimized with respect to the coefficients of the trigonometric series for stringer loads; then, the expressions for the stringer loads, shear flows, and ring bending moments are substituted into the resulting equation. This process yields a fourth-order finite-difference equation which can be solved for these trigonometric coefficients. The solution is then substituted back into the original expressions for stringer loads, shear flows, and ring moments to yield the desired distributions.

For convenience in application, the significant equations are collected in appendix A.

PERTURBATION LOAD SOLUTIONS CONCENTRATED PERTURBATION LOAD

Expression for stringer loads.—The concentrated perturbation load is shown in figure 3 (a); let P represent the magnitude of this load. Since the structure is uniform and infinitely long, half of the load goes into the portion of the structure to the right of the ring where the load is applied (ring i=0), and half goes to the left of this ring. Therefore, it can be seen from figure 3 (a) that, because of symmetry,

$$p_{ij} = -p_{-ij} \qquad (i \ge 1) \\ q_{ij} = q_{-i-i,j} \qquad (i \ge 0) \\ M(i,\phi) = -M(-i,\phi) \qquad (i \ge 0)$$
 (6)

Consider the right half of the structure, including the ring at i=0. The concentrated perturbation load gives rise to stringer loads which are circumferentially symmetric about stringer j=0 (see fig. 3 (a)). Thus the stringer load distribution can be represented by a series of the form

$$p_{ij} = \sum_{n=0}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} f_n(i) \cos nj\delta \tag{7}$$

where the notation $\sum_{n=0}^{\frac{m}{2} \text{ or } \frac{m-1}{2}}$ means that the summation is

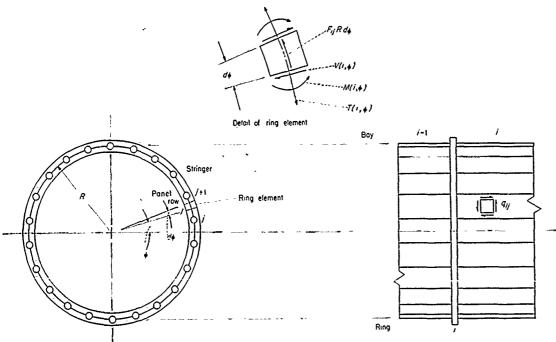


FIGURE 11.-Positive sense of quantities used in analysis.

rarried over n from n=0 to $n=\frac{m}{2}$ if m is even and to $n=\frac{m-1}{2}$ if m is odd:

Evaluation of $f_0(i)$, $f_1(i)$, and $f_n(0)$.—Suppose that equation (7) is multiplied by $\cos lj\delta$ and summed over j from 0 to m-1. This procedure yields

$$\sum_{i=0}^{m-1} p_{i,i} \cos lj\delta = \sum_{n=0}^{\frac{m}{2}} \int_{\pi}^{m-1} f_n(i) \sum_{j=0}^{m-1} \cos nj\hat{v} \cos lj\delta$$

The sum over j on the right-hand side is, for $0 \le n \le \frac{m}{2}$ and $0 \le l \le \frac{m}{2}$,

$$\sum_{j=0}^{m-1} \cos nj\delta \cos lj\delta = 0 \qquad (l \neq n)$$

$$= \frac{m}{2} \left(1 + \delta_{nk} + \delta_{n, \frac{m}{2}} \right) \qquad (l=n)$$

Thus the coefficients of the trigonometric series in equation (7) are

$$f_n(i) = \frac{2}{m\left(1 + \delta_n o + \delta_{n,\frac{m}{2}}\right)} \sum_{j=0}^{m-1} p_{ij} \cos nj\delta \tag{8}$$

It is desirable first of all to determine those values of $f_n(i)$ which can be found from consideration of the boundary conditions and of the overall equilibrium of the cylinder. Consider the equations of statics for the cylinder as a whole. Satisfaction of equilibrium in the longitudinal direction requires that the sum of the stringer loads at any ring station i must equal one-half of the applied load P. This condition is expressed as

$$\sum_{i=0}^{m-1} p_{ij} = \frac{P}{2}$$

For n=0, equation (8) gives

$$f_0(i) = \frac{1}{m} \sum_{j=0}^{m-1} p_{ij} = \frac{P}{2m}$$
 (9)

Moment equilibrium gives two equations, one of which is automatically satisfied because of the symmetry of the stringer load distribution around the cylinder. The other moment equation is

$$\sum_{j=0}^{m-1} p_{ij} R \cos j\delta = \frac{PR}{2}$$

For n=1, equation (8) is

$$f_{i}(i) = \frac{2}{m} \sum_{i=0}^{m-1} p_{ij} \cos j\delta = \frac{P}{m}$$
 (10)

On substituting the values of $f_0(i)$ and $f_1(i)$ given in equations (9) and (10), respectively, into equation (7), there results

$$p_{ij} = \frac{P}{2m} + \frac{P}{m} \cos j\delta + \sum_{n=2}^{\frac{m}{2} \text{ of } \frac{m-1}{2}} f_n(i) \cos nj\delta$$
 (11)

Consider now the boundary condition at ring i=0. The stringer loads here are

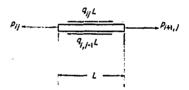
$$p_{0j} = \frac{P}{2} \delta_{0j}$$

and substitution of this expression into equation (8) yields

$$f_n(0) = \frac{P}{m\left(1 + \delta_{n0} + \delta_{n,\frac{m}{2}}\right)} \qquad \left(0 \le n \le \frac{m}{2}\right) \tag{12}$$

The equations of equilibrium and the boundary condition at i=0 have been used to obtain certain of the coefficients of the trigonometric series for stringer loads. The remainder of the coefficients are found by use of the principle of minimum complementary energy, and this is the next step in the solution.

Expressions for shear flows and ring bending moments.—In order to use the principle of minimum complementary energy, the shear flows and ring bending moments must be found in terms of the trigonometric coefficients $f_n(i)$. Shear flows are determined by the consideration of the equations of statics of a portion of any stringer j between two adjacent rings i and i+1. The forces on this free body are shown in sketch (a):



Sketch (a),

Equilibrium of these forces requires that

$$p_{i+1,j} - p_{ij} + (q_{ij} - q_{i,j-1})L = 0 (13)$$

Substitution of equation (11) into equation (13) yields

$$q_{i,j} - q_{i,j-1} = -\frac{1}{L} \sum_{n=2}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} [f_n(i+1) - f_n(i)] \cos nj\delta \qquad (14)$$

In order to find q_{ij} , replace j with a dummy index k and sum both sides of this equation over k from k=1 to k=j; that is, write

$$\sum_{k=1}^{J} (q_{ik} - q_{i,k-1}) = -\frac{1}{L} \sum_{n=2}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} [f_n(i+1) - f_n(i)] \sum_{k=1}^{J} \cos nk\delta$$

When the indicated summations over k have been carried out, the following equation is obtained:

(11)
$$q_{ij} - q_{i0} = -\frac{1}{L} \sum_{n=2}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} [f_n(i+1) - f_n(i)] \left[\frac{\sin n \left(j + \frac{1}{2} \right) \delta}{2 \sin \frac{n\delta}{2}} - \frac{1}{2} \right]$$

The term q_{s0} can be found from the condition that the total torque on the section is zero. The resulting expression for shear flows is

$$q_{ij} = -\sum_{n=2}^{\frac{m}{2}} \frac{\sum_{n=2}^{m-1} \frac{1}{2} f_n(i+1) - f_n(i)}{2L \sin \frac{n\delta}{2}} \sin n \left(j + \frac{1}{2}\right) \delta$$
 (15)

Bending moments are caused in each ring by a tangential loading which develops because of the difference in shear flow in the sheet on either side of the ring. The tangential load on ring i has the value

$$q_{ij} - q_{i-1,j} = -\sum_{n=2}^{\frac{m}{2}} \frac{\cot \frac{m-1}{2}}{2L \sin \frac{n\delta}{2}} \sin n \left(j + \frac{1}{2}\right) \delta \qquad (16)$$

In appendix B, this load is applied to a circular ring and the bending moment in the ring is derived. This procedure results in the following moment in ring i (see eq. (B9)):

$$M(i,\phi) = -\frac{\frac{m}{2} \text{ or } \frac{m-1}{2}}{\sum_{n=2}^{m} \frac{2\pi L}{2\pi L}} \Delta_{i,l} f_n(i) H_1(n,\phi)$$
 (17)

where

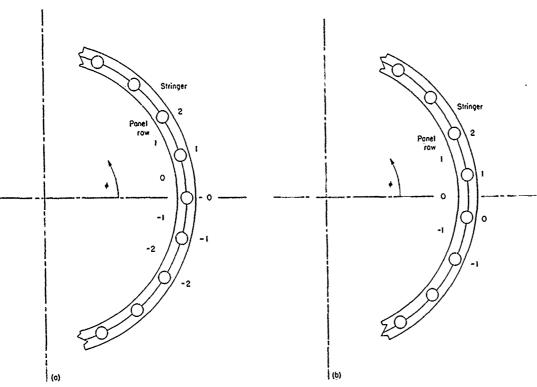
$$II_1(n,\phi) = \sum_{r=-\infty}^{\infty} \frac{\cos(rm+n)\phi}{(rm+n)^2[(rm+n)^2-1]}$$

The sign convention for the moment is illustrated in figure 11; the convention for measuring the angle ϕ is shown in figure 12 (a).

Energy analysis.—The stringer loads, shear flows, and ring bending moments have now been expressed in terms of the coefficients $f_n(i)$. The stringer loads are given in equation (11), the shear flows in equation (15), and the bending moments in equation (17). These equations are used in the minimization of the stress energy of the cylinder with respect to $f_n(i)$.

By virtue of the symmetry properties of this problem expressed in equations (6), the energy in the structure to the left of ring i=0 is the same as the energy to the right of ring i=0. Thus, equation (5) becomes

$$U = 2 \sum_{i=0}^{\infty} \sum_{j=0}^{m-1} \left[\frac{L}{6AE} (p_{ij}^2 + p_{ij}p_{i+1,j} + p_{i+1,j}^2) + \frac{R\delta L}{2Gl} q_{ij}^2 \right] + 2 \sum_{l=1}^{\infty} \int_{0}^{2\pi} \frac{R}{2EI} M^2(i,\phi) d\phi$$



(a) For concentrated and distributed perturbation-loads.

(b) For shear perturbation load,

FIGURE 12.-Conventions for angular coordinate \(\phi \).

Note that $M(0,\phi)$ is identically zero because there is no difference in shear flow across ring i=0 and, therefore, no tangential load acts on this ring.

Minimization of the stress energy with respect to $f_n(i)$ re-

sults in the following equation:

$$\frac{\partial U}{\partial f_{n}(i)} = 0$$

$$= \sum_{I=0}^{m-1} \left[\frac{L}{6AE} (p_{i+1, j} + 4p_{ij} + p_{i-1, j}) \frac{\partial p_{ij}}{\partial f_{n}(i)} + \frac{R\delta L}{Gi} \left(q_{ij} \frac{\partial q_{ij}}{\partial f_{n}(i)} + q_{i-1, j} \frac{\partial q_{i-1, j}}{\partial f_{n}(i)} \right) \right] + \int_{0}^{2\pi} \frac{R^{j}}{EI} \left[M(i+1, \phi) \frac{\partial M(i+1, \phi)}{\partial f_{n}(i)} + M(i, \phi) \frac{\partial M(i, \phi)}{\partial f_{n}(i)} + M(i-1, \phi) \frac{\partial M(i-1, \phi)}{\partial f_{n}(i)} \right] d\phi \tag{18}$$

The coefficients $f_0(i)$ and $f_1(i)$ are known already for all values of i, and $f_n(0)$ is known for $0 \le n \le \frac{m}{2}$. Equation (18)

therefore needs only to be considered for $i \ge 1$ and $n \ge 2$. The expressions for the stringer loads, shear flows, and ring bending moments are substituted into equation (18). Then the following definite sums are needed (these can be obtained by the procedure outlined in ref. 7):

$$\sum_{j=0}^{m-1} \cos nj\delta = 0 \qquad (0 < n < m) \tag{19}$$

and for the integers n and l restricted to the range $1 \le n \le \frac{m}{2}$ and $1 \le l \le \frac{m}{2}$.

$$\sum_{j=0}^{m-1} \cos lj\delta \cos nj\delta = 0 \qquad (l \neq n)$$

$$= \frac{m}{2} \left(1 + \delta_{n, \frac{m}{2}} \right) \qquad (l=n)$$
(20)

and

$$\sum_{j=0}^{m-1} \sin l \left(j + \frac{1}{2} \right) \delta \sin n \left(j + \frac{1}{2} \right) \delta = 0 \qquad (l \neq n)$$

$$= \frac{m}{2} \left(1 + \delta_{n, \frac{m}{2}} \right) \quad (l = n)$$
(21)

The following definite integral, which is derived in appendix C, is also needed:

$$\int_{0}^{2\pi} II_{1}(n,\phi) II_{1}(l,\phi) d\phi = 0 \qquad (l \neq n)$$

$$= S_{n}\pi \left(\frac{1 + \delta_{n,\frac{m}{2}}}{n} \right) \qquad (l = n)$$

$$= \left(\frac{1 + \delta_{n,\frac{m}{2}}}{n} \right) \qquad (l = n)$$

$$= \left(\frac{1 + \delta_{n,\frac{m}{2}}}{n} \right) \qquad (l = n)$$

$$= \left(\frac{1 + \delta_{n,\frac{m}{2}}}{n} \right) \qquad (l = n)$$

where

$$S_{n'} = \sum_{r=-\infty}^{\infty} D_{rn}^{2} = \sum_{r=-\infty}^{\infty} \frac{1}{(rm+n)^{4} [(rm+n)^{2}-1]^{2}}$$

and where n and l are restricted to $2 \le n \le \frac{m}{2}$ and $2 \le l \le \frac{m}{2}$.

A closed form of S_n is presented in appendix C but the series form converges so rapidly that it is usually more convenient than the closed form for use in calculations.

After substitution of the expressions for stringer loads, shear flows, and ring moments into equation (18), the use of these definite sums (19), (20), and (21), and definite integral (22) results in the following equations which express the condition of minimum stress energy:

For i=1,

$$f_n(3) + 2\gamma_n f_n(2) + (2\beta_n - 1)f_n(1) + 2(\gamma_n + 1)f_n(0) = 0 \quad (23a)$$
 and, for $i \ge 2$,

$$f_{\rm n}(i+2) + 2\gamma_{\rm n}f_{\rm n}(i+1) + 2\beta_{\rm n}f_{\rm n}(i) + 2\gamma_{\rm n}f_{\rm n}(i-1) + f_{\rm n}(i-2) = 0 \eqno(23b)$$

where

$$1 - \frac{3}{2} \frac{B\delta^{2}}{\sin^{2} \frac{R\delta^{2}}{2}}$$

$$\gamma_{n} = -2 + \frac{1}{12CS_{n}}$$

$$4 + 3 - \frac{B\delta^{2}}{\sin^{2} \frac{n\delta}{2}}$$

$$\beta_{n} = 3 + \frac{1}{12CS_{n}}$$

$$B = \frac{E t'}{G} \frac{R^{2}}{l} \frac{R^{2}}{L^{2}}$$

$$C = \frac{t'R^{6}}{lT_{3}}$$

Solution of finite-difference equation.—Equation (23b) is a fourth-order finite-difference equation with constant coefficients. (Note that the symbol i represents the index of the rings and bays and should not be confused with the usual notation for $\sqrt{-1}$ which never appears in this report.) Equation (23b) corresponds exactly with equation (24) of reference 8. The general solution is presented on pages 23 to 26 of reference 8 and on pages 28 and 29 of reference 9. It may be written as

$$f_{n}(i) = (\pm e^{-\psi_{n}})^{t} [\alpha_{1n}\Lambda_{1n}(i) + \alpha_{2n}\Lambda_{2n}(i)] + (\pm e^{\psi_{n}})^{t} [\alpha_{3n}\Lambda_{1n}(i) + \alpha_{4n}\Lambda_{2n}(i)] \quad (n \ge 2)$$
 (24)

where the upper sign is used when $\gamma_n < 0$ and the lower sign when $\gamma_n > 0$. The values of Λ are as follows:

For
$$D_n = \frac{2(\beta_n - 1)}{\gamma_n^2} > 1$$
,

$$\Lambda_{1n}(i) = \cos i\chi_n$$

$$\Lambda_{2n}(i) = \sin i\chi_n$$

where

$$\chi_{n} = \frac{1}{2} \cos^{-1} \left[\frac{\beta_{n} - 1}{2} - \sqrt{\left(\frac{\beta_{n} + 1}{2} \right)^{2} - \gamma_{n}^{2}} \right]$$

For $D_n < 1$,

$$h_{1n}(i) = \cosh i \chi_n$$

$$\Lambda_{2n}(i) = \sinh i\chi_n$$

where

$$\chi_{\mathbf{n}} = \frac{1}{2} \cosh^{-1} \left[\frac{\beta_{\mathbf{n}} - 1}{2} - \sqrt{\left(\frac{\beta_{\mathbf{n}} + 1}{2}\right)^2 - \gamma_{\mathbf{n}}^2} \right]$$

For $D_n = 1$,

$$\Lambda_{1a} = 1$$

$$\Lambda_{2a} = i$$

In the inverse trigonometric and hyperbolic functions, the principal values are used. The argument ψ_n of the exponential function is given by the positive branch of

$$\psi_{n} = \frac{1}{2} \cosh^{-1} \left[\frac{\beta_{n} - 1}{2} + \sqrt{\left(\frac{\beta_{n} + 1}{2}\right)^{2} - \gamma_{n}^{2}} \right]$$

At a large longitudinal distance from the applied load, the stringer loads should approach the elementary distribution given by the first-two terms of equation (11); consequently, for $n \ge 2$, $f_n(i)$ approaches zero as i approaches infifity. The first term on the right-hand side of equation (24) satisfies this condition; however, the second term does not and, hence, must be omitted. The solutions, then, that are compatible with the boundary conditions at infinity are:

$$f_n(i) = \zeta_n^{-1} \left[\alpha_{1n} \Lambda_{1n}(i) + \alpha_{2n} \Lambda_{2n}(i) \right] \qquad (n \ge 2)$$
 (25)

where

Now the arbitrary constants α_{1n} and α_{2n} are determines? The first, α_{1n} , is obtained immediately. Substitution of i=0 into equation (25) and use of equation (12) to evaluate $f_n(0)$ yields

$$f_n(0) = \alpha_{1n} = \frac{P}{m\left(1 + \delta_{n,\frac{m}{2}}\right)} \qquad (n \ge 2)$$
 (26)

Substitution of equations (26) and (25) into the boundary equation (23a) yields

$$\alpha_{2n} = \frac{\Theta_{1n} + 2(\gamma_n + 1)}{\Theta_{2n}} \frac{P}{m\left(1 + \delta_{n,\frac{m}{2}}\right)}$$

where

$$O_{sn} = \zeta_n^3 \Lambda_{sn}(3) + 2\gamma_n \zeta_n^2 \Lambda_{sn}(2) + (2\beta_n - 1)\zeta_n \Lambda_{sn}(1) \qquad (s = 1, 2)$$

The solution for the concentrated perturbation load is now complete since the coefficients $f_n(i)$ are completely defined and may be substituted into equation (11) to give the stringer loads. The shear flows can be found from equation (15); however, once the stringer loads are known, it is simpler to calculate the shear flows by the use of the equations of statics. Because of symmetry, the shear flows in shear

panels adjacent to stringer j=0 are given by

$$q_{i0} = -q_{i,-1} = \frac{p_{i0} - p_{i+1,0}}{2L}$$

All the other shear flows can be found by the use of equation (13). If desired, the moment distribution in the rings can be computed from equation (17) and the thrust and transverse shear in the rings can be found from the formulas given in appendix B.

DISTRIBUTED PERTURBATION LOAD

Expression for stringer loads.—The distributed perturbation load is shown in figure 3 (b); let S represent the magnitude of the total force distributed along stringer j=0 between rings i=0 and i=1. From figure 3 (b) it is seen that

$$p_{ij} = -p_{-i+1,j} \qquad \qquad (i \ge 1) \quad (27a)$$

$$q_{ij} = q_{-i,j}$$
 $(i \ge 1)$ (27b)

$$M(i,\phi) = -M(-i+1,\phi)$$
 $(i \ge 1)$ (27c)

At ring i=1 and to the right of this ring, the stringer loads can be represented by a trigonometric series of exactly the same form as equation (7)

$$p_{ij} = \sum_{n=0}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} f_n(i) \cos nj\delta$$
 (28)

except now $i \ge 1$, and the coefficients $f_n(i)$ are different from those obtained for the preceding case of the concentrated load.

Evaluation of $f_0(i)$ and $f_1(i)$.—As in the preceding case, the first two coefficients $f_0(i)$ and $f_1(i)$ can be obtained from the equations of statics, and the results are the same as before. Equation (28) becomes

$$p_{ij} = \frac{S}{2m} + \frac{S}{m} \cos j\delta + \sum_{n=2}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} f_n(i) \cos nj\delta \quad (i \ge 1) \quad (29)$$

With the concentrated perturbation load, all the coefficients $f_n(0)$ were easily found because the stringer load distribution at-ring station i=0 was known. Here no such distribution is known. In order to determine the boundary condition at bay i=0, the effect of the distributed perturbation load-on the equilibrium of portions of stringers in this bay must be investigated.

Expressions for shear flows and ring bending moments.— Away from bay i=0 the shear flows and ring bending moments are of the same form as for the concentrated load. The following expression for the shear flows is obtained by use of equation (13):

$$q_{ij} = -\sum_{n=2}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} \frac{f_{2}(i+1) - f_{n}(i)}{2L \sin \frac{n\delta}{2}} \sin n \left(j + \frac{1}{2}\right) \delta \quad (i \ge 1) \quad (30)$$

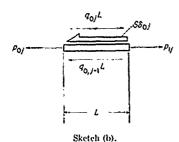
The ring bending moments are obtained in appendix B as

$$M(i,\phi) = -\sum_{n=2}^{\frac{m}{2}} \frac{n-1}{2\pi L} \frac{R^2 m}{2\pi L} \Delta_{il} f_n(i) H_1(n,\phi) \qquad (i \ge 2) \quad (31)$$

where

$$II_1(n,\phi) = \sum_{r=-\infty}^{\infty} \frac{\cos{(rm+n)\phi}}{(rm+n)^2[(rm+n)^2-1]}$$

The applied force in bay i=0 may be written as $S\delta_{0j}$. Consider, now, the equilibrium of a portion of any stringer j, between ring i=0 and ring i=1. The forces on this free body are shown in sketch (b):



Equilibrium of these forces requires that

$$p_{1j} - p_{0j} + (q_{0j} - q_{0,j-1})L - S\delta_{0j} = 0$$

Because of the antisymmetry property expressed in equation (27a), the equilibrium equation becomes

$$2p_{1j} + (q_{0j} - q_{0,j-1})L - S\delta_{0j} = 0$$
 (32)

It is convenient, now, to expand the Kronecker delta δ_{ij} in a finite trigonometric series,

$$\delta_{0j} = \sum_{n=0}^{\frac{m}{2} \text{ or } m - 1 \atop n = 0} d_n \cos nj\delta \tag{33}$$

Multiplying through by $\cos lj\delta$ and summing over j from 0 to m-1 yields the trigonometric coefficients d_n . The result is

$$d_{n} = \frac{2}{m\left(1 + \delta_{n0} + \delta_{n,\frac{m}{2}}\right)} \tag{34}$$

Substitution of the expression for stringer loads (equation (29)) and the trigonometric expansion for δ_{0f} (equation (33)) into the equilibrium equation (32) yields

$$q_{0j} - q_{0,j-1} = \frac{1}{L} \sum_{n=2}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} [Sd_n - 2f_n(1)] \cos nj\delta$$

In order to find q_{0j} , this equation can be treated in the same manner as equation (14); that is, replace j by a dummy index k, sum from k=1 to k=j, and then use the condition that the total torque on a cross section in bay i=0 must be zero. This procedure results in the following expression for the shear flows in bay i=0:

$$q_{0j} = \sum_{n=2}^{\frac{m}{2}} \frac{o^{r} \prod_{j=1}^{m-1} \frac{1}{2} S d_n - f_n(1)}{L \sin \frac{n\delta}{2}} \sin n \left(j + \frac{1}{2} \right) \delta$$
 (35)

The expression for the bending moment in rings i=1 and i=0 is yet to be found, as this expression differs from that for the moment in the rest of the rings given in equation (31). The moment in ring i=0 is the same in ranguitude as that in ring i=1 but opposite in sign. The tangential loading on ring i=1 is given by

$$q_{ij} - q_{0j} = -\frac{\frac{m}{2} \sum_{n=2}^{\text{or } \frac{m-1}{2}} f_n(2) - 3f_n(1) + Sd_n}{2L \sin \frac{n\delta}{2}} \sin n \left(j + \frac{1}{2}\right) \delta$$

By analogy with equations (16) and (17), then, the bending moment in ring i=1 can be written as

$$M(1,\phi) = -\sum_{n=2}^{\frac{m}{2}} \frac{n-1}{2\pi L} \frac{R^2 m}{2\pi L} \left[\int_{n} (2) - 3 \int_{n} (1) + S d_{n} \right] H_{1}(n,\phi)$$
 (36)

All the stringer loads, shear flows, and ring bending moments have now been expressed in terms of the coefficients $f_n(i)$. The stringer loads are given in equation (29), the shear flows in equations (30) and (35), and the ring moments in-equations (31) and (36). The next step in the analysis is the substitution of these expressions into the equation obtained from minimization of the stress energy of the cylinder with respect to $f_n(i)$.

Energy analysis.—By virtue of the symmetry proporties in this problem given in equations (27), the energy in the structure to the right of bay i=0 equals the energy to the left of this bay. Equation (5) for the stress energy can be written

$$U = \sum_{j=0}^{m-1} \left(\frac{L}{6AE} p_{ij}^2 + \frac{R\delta L}{2Gt} q_{0j}^2 \right) + 2 \sum_{i=1}^{m} \sum_{j,k=0}^{m-1} \left[\frac{L}{6AE} (p_{ij}^2 + p_{ij}p_{i+1,j} + p_{i+1,j}^2) + \frac{R\delta L}{2Gt} q_{ij}^2 \right] + 2 \sum_{i=1}^{m} \int_{0}^{2\pi} \frac{R}{2EI} M^2(i,\phi) d\phi$$
(37)

Minimization of the stress energy with respect to $f_n(i)$ results in the following-equations:

$$\frac{\partial U}{\partial f_n(1)} = 0 = \sum_{j=0}^{m-1} \left[\frac{L}{6iL} (3p_{1i} + p_{2i}) \frac{\partial p_{1i}}{\partial f_n(1)} + \frac{R\delta L}{2\tilde{u}\tilde{u}} \left(2q_{1j} \frac{\partial q_{1j}}{\partial f_n(1)} + q_{2j} \frac{\partial q_{0j}}{\partial f_n(1)} \right) \right] + \int_0^{2\pi} \frac{R}{EI} \left[M(2,\phi) \frac{\partial M(2,\phi)}{\partial f_n(1)} + M(1,\phi) \frac{\partial M(1,\phi)}{\partial f_n(1)} \right] d\phi \quad (38)$$

and

$$\frac{\partial U}{\partial f_{n}(i)} = 0 = \sum_{j=0}^{m-1} \left[\frac{L}{6AE} \left(p_{i+1,j} + 4p_{ij} + p_{i-1,j} \right) \frac{\partial p_{ij}}{\partial f_{n}(i)} + \frac{R\delta L}{6i} \left(q_{ij} \frac{\partial q_{ij}}{\partial f_{n}(i)} + q_{i-1,j} \frac{\partial q_{i-1,j}}{\partial f_{n}(i)} \right) \right] + \int_{0}^{2\pi} \frac{R}{EI} \left[M(i+1,\phi) \frac{\partial M(i+1,\phi)}{\partial f_{n}(i)} + M(i+1,\phi) \frac{\partial M(i+1,\phi)}{\partial f_{n}(i)} \right] d\phi \qquad (i \ge 2)$$

$$(39)$$

Note that equation (39) is the same as equation (18), except that equation (39) is valid only for $i \ge 2$.

The stringer loads, shear flows, and ring moments are substituted into equations (38) and (39), and then the definite sums and definite integral derived in the preceding section are used to simplify these equations. After simplification, the following equations result: For i=1.

$$f_n(3) + (2\gamma_n - 1) f_n(2) + 2(\beta_n - \gamma_n) f_n(1) = Sd_n\left(\frac{\beta_n - 4\gamma_n - 2}{3}\right). \tag{40a}$$

For i=2.

$$f_n(4) + 2\gamma_n f_n(3) + 2\beta_n f_n(2) + (2\gamma_n - 1) f_n(1) = -Sd_n$$
 (40b)

For $i \ge 3$,

$$f_n(i+2) + 2\gamma_n f_n(i+1) + 2\beta_n f_n(i) + 2\gamma_n f_n(i-1) + f_n(i-2) = 0$$
(40c)

Solution of finite-difference equation.—Equation (40c) is the same as equation (23b); therefore, the solution to equation (40c) is

$$f_n(i) = \int_{R} \{\alpha_{1n} \Lambda_{1n}(i) + \alpha_{2n} \Lambda_{2n}(i)\} \quad (n \ge 2)$$
 (41)

which is the same as equation (25) except for the values of the arbitrary constants α_{1n} and α_{2n} . These constants are found by the substitution of the solution (41) into equations (40a) and (40b). This procedure yields two simultaneous algebraic equations in α_{1n} and α_{2n} , and their solution gives

$$\alpha_{1n} = \frac{\Gamma_{1n} \frac{\beta_n - 4\gamma_n - 2}{3} + \Omega_{2n}}{\Omega_{1n} \Gamma_{2n} - \Gamma_{1n} \Omega_{2n}} \frac{2S}{m(1 + \delta_{n, \frac{m}{2}})}$$

$$\alpha_{2n} = \frac{\Omega_{1n} + \Gamma_{1n} \frac{\beta_n - 4\gamma_n - 2}{3}}{\Omega_{1n} \Gamma_{2n} - \Gamma_{1n} \Omega_{2n}} \frac{2S}{m(1 + \delta_{n, \frac{m}{2}})}$$

where d_n , the coefficient in the trigonometric series for the Kronecker delta δ_{0l} , has been replaced by its value as given in equation (34), and where the Ω 's and Γ 's are given by

$$\Omega_{In} = \zeta_n^3 \Lambda_{In}(3) + (2\gamma_n - 1)\zeta_n^2 \Lambda_{In}(2) + 2(\beta_n - \gamma_n)\zeta_n \Lambda_{In}(1)$$

$$(s = 1, 2) \quad (42a)$$

$$\Gamma_{ss} = \int_{s}^{4} \Lambda_{sn}(4) + 2\gamma_{s} \int_{s}^{3} \Lambda_{sn}(3) + 2\beta_{s} \int_{s}^{4} \Lambda_{sn}(2) + (2\gamma_{s} - 1) \int_{s} \Lambda_{sn}(1) \qquad (s = 1, 2)$$

$$(42b)$$

The coefficients $f_n(i)$ are now defined for the distributed perturbation load and-may be substituted into equation (29) to give the stringer loads. The shear flows can be found from equations (30) and (35), but, again, once the stringer loads are known, shear flows can easily be found by use of the equations of statics. The shear flow in the panels adjacent to stringer j=0 can be found by considering symmetry: In bay i=0

$$q_{\infty} = -q_{0,-1} = \frac{S - 2p_{10}}{2L}$$

and, outside of bay i=0,

$$q_{i,-1} = \frac{p_{i,-1} - p_{i+1,0}}{2L}$$
 $(i \ge 1)$

The other shear flows are found from equation (13), as before. If desired, the ring moments can be obtained from equations (31) and (36) and the ring thrust and transverse shear can be found from the equations given in appendix B.

SHEAR PERTURBATION LOAD

Expression for stringer loads.—The shear perturbation load is shown in figure 3 (c). The magnitude of the load per unit length applied along the stringers and rings bordering shear panel (0,0) will be represented by Q. From figure 3(c) it is seen that the longitudinal symmetry properties in this case are the same as those for the case of the distributed perturbation load given by equation (27).

The shear perturbation load is self-equilibrating and gives rise to stringer loads which are antisymmetric about panel row j=0. For $i\ge 1$, the stringer loads may be represented by

$$p_{ij} = \sum_{n=1}^{\frac{m}{2} \text{ of } n-1} f_n(i) \sin n \left(j - \frac{1}{2}\right) \delta \tag{43}$$

where the coefficients $f_n(i)$ are different from those in the two preceding cases. The term corresponding to n=1 vanishes because it represents an elementary bending stringer-load distribution, and the shear perturbation load does not require this distribution for overall equilibrium.

Expressions for shear flows and ring bending moments. – None of the coefficients $f_n(i)$ in the trigonometric series (43) can be found from the equations of statics. Furthermore, the boundary condition at bay i=0 must be determined from a consideration of the effect that the shear perturbation load

has on the equilibrium of the portions of stringers in bay i=0 and on the bending moment in the rings bounding this bay. Thus the energy approach must be used-immediately and the first step in this approach is to write the shear flows and ring moments in terms of $f_n(i)$, the coefficients f the trigonometric series for the stringer loads, equation (43).

Outside of bay i=0, the satisfaction of the equations of statics for the portions of stringers between adjacent rings yields equation (13), the same as in the two preceding cases. Substituting equation (43) for the stringer loads into the equilibrium equation (13) and following the same procedure used to obtain equation (15) yields the expression for the shear flows due to the shear perturbation load:

$$q_{ij} = \sum_{n=2}^{\frac{m}{2}} \frac{\int_{n} \frac{m-1}{2} \int_{n} (i+1) - \int_{n} (i)}{2L \sin \frac{n\delta}{2}} \cos nj\delta \qquad (i \ge 1) \qquad (44)$$

The-tangential loadings on the rings to the right of ring

$$q_{ij} - q_{i-1,j} = \sum_{n=2}^{\frac{m}{2} \text{ of } \frac{m-1}{2}} \frac{\Delta_{ii} f_n(i)}{2L \sin \frac{n\delta}{2}} \cos nj\delta$$

In appendix B this load is applied to a circular ring and the following expression for the moment in the ring is obtained (see eq. (B13)):

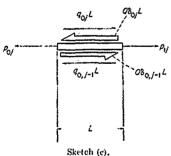
$$M(i,\phi) = -\frac{\sum_{n=2}^{m} \frac{m-1}{2}}{\sum_{n=2}^{m} \frac{R^{2}m}{2\pi L}} \Delta_{tt} f_{n}(i) H_{2}(n,\phi) \qquad (i \ge 2) \quad (45)$$

where

$$II_2(n,\phi) = \sum_{r=-\infty}^{\infty} (-1)^r \frac{\sin(rm+n)\phi}{(rm+n)^2[rm+n)^2-1}$$

The convention for measuring the angle ϕ here is a little different than before and is illustrated in figure $12/\tilde{\rho}h$.

Now, the shear flows in bay i=0 and the bending moments in the rings bordering bay i=0 must be found. Consider the shear flows in this central bay. The shear perturbation loading applied at bay i=0 may be written $Q\delta_{0j}$. Then the forces on the portion of any stringer j between ring i=0 and ring i=1 are as shown in sketch (c):



Equilibrium of these forces requires that

$$p_{1j} - p_{0j} + (q_{0j} - q_{0,j-1})L + Q(\delta_{0,j-1} - \delta_{0j})L = 0$$

Because of the antisymmetry property, equation (27a), the equation of equilibrium becomes

$$2p_{3,i} + (q_{0,i} - q_{0,i-1})L + Q(\delta_{0,i-1} - \delta_{0,i})L = 0$$
 (46)

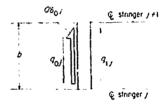
The substitution of the stringer loads (equation (43)) Into the equilibrium equation (46), and the introduction of the trigonometric expansion for the Kronecker delta δ_{0j} (equation (33)) yields the following equation:

$$\begin{aligned} q_{0j} - q_{0,j-1} &= -\frac{1}{L} \sum_{n=2}^{\frac{m}{2}} \frac{2f_n(1) \sin n \left(j - \frac{1}{2}\right) \delta -}{2d_1 \left[\cos (j-1)\delta - \cos j\delta\right]} \\ Qd_1 \left[\cos (j-1)\delta - \cos j\delta\right] &= \sum_{n=2}^{\frac{m}{2}} Qd_n \left[\cos n (j-1)\delta - \cos n j\delta\right] \end{aligned}$$

Now q_{0j} can be found by replacing j with a dummy index k, summing over k from k=1 to k=j, and using the condition that the torque on a cross section within bay i=0 balances the applied torque. This procedure results in the following equation for the shear flow in the central-bay:

$$q_{0j} = Qd_0 + Qd_1 \cos j\delta + \sum_{n=2}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} \left(\frac{\int_{\pi} (1)}{L \sin \frac{n\delta}{2}} + Qd_n \right) \cos nj\delta \quad (47)$$

Consider the bending moment in rings i=1 and i=0. The moment in ring i=0 is identical in magnitude to the moment in ring i=1 but of opposite sign. The tangential loading per unit are length on the portion of ring i=1 between stringer j and stringer j+1 is illustrated in sketch (d):



Sketch (d),

When these tangential loads are added and the series expansions for q_{0j} , q_{1j} , and δ_{0j} are introduced, the total load per unit are length on ring i=1 is given by

$$q_{1j}-q_{0j}+Q\delta_{0j}=\sum_{n=2}^{\frac{m}{2}}\sum_{n=2}^{m-1}\frac{f_{n}(2)-3f_{n}(1)}{2I_{n}\sin\frac{n\delta}{2}}\cos nj\delta$$

By analogy with equations (16) and (17) the bending moment in ring i=1 is

$$M(1,\phi) = -\sum_{n=2}^{\frac{m}{2} \text{ or } m-1} \frac{P^2 m}{2\pi L} \left[f_n(2) - 3f_n(1) \right] H_2(n,\phi) \tag{48}$$

Expressions for stringer loads, shear flows, and ring moments have been written in terms of the coefficients $f_n(i)$. The stringer loads are given in equation (43), the shear flows in equations (44) and (47), and ring moments

in equations (45) at 1 (48). These expressions are ready to be substituted into the equation which results from minimizing the stress energy with respect to $f_n(i)$.

Energy analysis.—Because the longitudinal symmetry relations which-exist-for the distributed perturbation load, equations (27), also exist in the case of the shear perturbation load, the stress-energy expression used in the distributed-load problem can-be used here. The expressioas obtained on minimizing this-stress energy, equations (38) and (39), are also applicable here. Consequently, the stringer loads, shear flows, and ring moments just derived are substituted into equations (38) and (39). At this stage in the two preceding cases, certain definite sums and a definite integral were introduced to simplify the equations. A similar procedure is followed here.

The definite sums which are of interest are

$$\sum_{i=0}^{m-1} \sin n \left(j - \frac{1}{2} \right) \delta = 0$$

and for the integers n and l restricted to the range $1 \le n \le \frac{m}{2}$ and $1 \le l \le \frac{m}{2}$

$$\sum_{l=0}^{m-1} \cos lj\delta \cos nj\delta = 0 \qquad (l \neq n)$$

$$=\frac{m}{2}\left(1+\delta_{n,\frac{m}{2}}\right) \qquad (l=n)$$

and

$$\sum_{j=0}^{m-1} \sin l \left(j - \frac{1}{2} \right) \delta \sin n \left(j - \frac{1}{2} \right) \delta = 0 \qquad (l \neq n)$$

$$= \frac{m}{2} \left(1 + \delta_{n, \frac{m}{2}} \right) \qquad (l = n)$$

The required definite integral, which is derived in appendix C, is

$$\int_{0}^{2\pi} II_{2}(n,\phi)II_{2}(l,\phi)d\phi = 0 \qquad (l \neq n)$$

$$=S_n\pi\left(1+\delta_{n,\frac{m}{n}}\right) \qquad (l=n)$$

where n and l are restricted to $2 \le n \le \frac{m}{2}$ and $2 \le l \le \frac{m}{2}$. After simplification the following equations result: For i=1.

$$f_n(3) + (2\gamma_n - 1) f_n(2) + 2(\beta_n - \gamma_n) f_n(1)$$

$$= -2LQ d_n \left(\frac{\beta_n - 4\gamma_n - 11}{3}\right) \sin \frac{n\delta}{2}$$
(49a)

For i=2,

$$f_n(4) + 2\gamma_n f_n(3) + 2\beta_n f_n(2) + (2\gamma_n - 1) f_n(1) = 0$$
 (49b)

For $i \ge 3$,

$$f_n(i+2) + 2\gamma_n f_n(i+1) + 2\beta_n f_n(i) + 2\gamma_n f_n(i-1) + f_n(i-2) = 0$$
(49c)

Solution of finite-difference equation.—Equation (49c) is the same finite-difference equation for which the solution is written in the two preceding sections. Substitution of this solution, equation (41), and equation (49a) and (49b)

gives two simultaneous algebraic equations for α_1 , and α_{2n} , the arbitrary constants. Solution of this system yields

$$\alpha_{1n} = -\frac{\Gamma_{2n} \frac{\beta_n - 4\gamma_n - 11}{3} \sin \frac{n\delta}{2}}{\Omega_{1n} \Gamma_{2n} - \Gamma_{1n} \Omega_{2n}} \frac{4QL}{m (1 + \delta_{n, \frac{m}{2}})}$$

$$\alpha_{2n} = \frac{\Gamma_{1n} \frac{\beta_n - 4\gamma_n - 11}{3} \sin \frac{n\delta}{2}}{\Omega_{1n} \Gamma_{2n} - \Gamma_{1n} \Omega_{2n}} \frac{4QL}{m(1 + \delta_{n, \frac{m}{2}})}$$

The Ω 's and Γ 's in this case are precisely the same as in the preceding case of the distributed perturbation load; Ω_{in} is given by equation (42a) and Γ_{in} by equation (42b).

With the coefficients $f_n(i)$ known for the shear perturbation load, the stringer loads are obtained from equation (43) and the shear flows can be found from equations (44) and (47). For panel row j=0, the shear flow equations become

$$q_{i0} = \sum_{n=2}^{\frac{m}{2}} \sum_{n=2}^{c} \frac{n-1}{2} \frac{f_n(i+1) - f_n(i)}{2L \sin \frac{n\delta}{2}} \qquad (i \ge 1)$$

and

$$q_{00} = \frac{3Q}{m} + \sum_{n=2}^{\frac{m}{2} \text{ or } \frac{n-1}{2}} \left[\frac{f_n(1)}{L \sin \frac{n\delta}{2}} + m \frac{2Q}{(1+\delta_{n,\frac{n}{2}})} \right]$$

When the shear flows in panel row j=0-are known, it is simpler to compute the remainder of the shear flows by use of the equations of statics rather than equations (44) and (47). In shear panels (0,1) and (0, -1) adjacent to the loaded panel, the shear flow is given by

$$q_{01} = q_{01} - q_{02} - 2p_{11} + QL$$

All the other shear flows are found by use of equation (13). If desired, the ring bending moments can be found from equations (45) and (48) and the ring thrust and transverse shear can be calculated from the formulas given in appendix B.

LIMITING CASE OF RIGID RINGS

If the ring bending stiffness is allowed to increase indefinitely, the rings approach complete rigidity in bending, the parameter C approaches zero, and a considerable simplification results. For this limiting case, equations (23) for the concentrated perturbation load reduce to

$$f_n(i+1) - 2\frac{B_n}{A_n} f_n(i) + f_n(i-1) = 0$$
 $(i \ge 1)$ (50)

where

$$A_n = 3B\delta^2 - 1 + \cos n\delta$$

$$B_n = 3B\delta^2 + 2(1 - \cos n\delta)$$

This can be shown easily by multiplying equations (23) through by C and allowing C to approach 0. Equation (50) is a second-order finite-difference equation with constant coefficients. The same equation, together with its general

solution, is given in reference 9, page 31. 'The solution compatible with the boundary conditions at infinity can be written as

$$f_n(i) = \alpha_n (\pm e^{-\lambda_n})^i \tag{51}$$

where

$$\cosh \lambda_n = \frac{|B_n|}{|A_n|}$$

and where the upper sign is taken when $A_n > 0$ and the lower sign when $A_n < 0$.

The arbitrary constant α_n is determined by evaluating the solutions, equation (51), for i=0 and introducing the value of $f_n(0)$ given in equation (12). The result is identical to α_{1n} given in equation (26)

$$\alpha_n = \frac{P}{m\left(1 + \delta_{n, \frac{m}{2}}\right)} \qquad (n \ge 2)$$

Equations (11) and (15), the expressions for stringer loads and shear flows, respectively, used before in the case of the concentrated perturbation load are still valid. The substitution into these expressions of the solution (51) with the constant α_n as found above yields the stringer loads and shear flows due to a concentrated perturbation load when the rings can be considered rigid.

For the case of the distributed perturbation load, equations (40) reduce in the limit to

$$(-A_n)f_n(2) + (2B_n + A_n)f_n(1) = 3B\delta^2 S d_n$$

$$f_n(i+1) - 2\frac{B_n}{A_n}f_n(i) + f_n(i-1) = 0 \qquad (i \ge 2)$$

The arbitrary constant α_s in the solution (51) is

$$\alpha_n = \frac{6B\delta^2}{A_n(\pm e^{-\lambda_n} + 1)} \frac{S}{m(1 + \delta_{n, \frac{m}{2}})}$$

For the shear perturbation load, equations (49) reduce to

$$(-A_n) f_n(2) + (2B_n + A_n) f_n(1) = -6LQ d_n B \delta^2 \sin \frac{n\delta}{2}$$
$$f_n(i+1) - 2 \frac{B_n}{A_n} f_n(i) + f_n(i-1) = 0 \qquad (i \ge 2)$$

The solution is again equation (51) and α_s becomes

$$\alpha_n = \frac{12B\delta^2 \sin \frac{n\delta}{2}}{A_n (\pm e^{-\lambda_n} + 1) \frac{m}{m} \frac{1+\delta_n \frac{m}{2}}{1+\delta_n \frac{m}{2}}}$$

CONCLUDING REMARKS

A method is presented for the stress analysis of circular semimonocoque cylinders with cutouts. It is most accurate in problems where the cutout is located far from external restraints. The loading may be any combination of torsion, bending, shear, or axial load. Other loadings are permissible if the stress distribution in the cylinder without a cutou known.

The method of analysis is based on the superposition of certain perturbation stress distributions to give the effects of the cutout on the stress distribution which would exist in the cylinder without a cutout. The equations for the three necessary perturbation stress distributions are derived in this report, and tables of coefficients calculated from these equations are presented for a wide range of structural properties. Ring bending flexibility is taken into account in the tables. The tables refer to a structure having 36 stringers, but they can be used for cylinders having any number of stringers by redistribution of the actual stringer area into 36 flectitions stringers. Sample calculations utilizing the tables of coefficients are presented to illustrate the analytical procedure.

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APPENDIX A

SUMMARY OF SIGNIFICANT EQUATIONS

The formulas and parameters required for computing the stress distribution due to concentrated, distributed, and shear-perturbation loads are collected in this appendix for reference.

STRINGER LOADS

Concentrated perturbation load (see fig. 3-(a)):

$$p_{ij} = \frac{P}{2m} + \frac{P}{m} \cos j\delta + \sum_{k=0}^{m-1} f_{n}(i) \cos nj\delta \qquad (i \ge 0)$$

where-P is-the-applied load.

Distributed perturbation load (see fig. 3 (b)):

$$p_{ij} = \frac{S}{2m} + \frac{S}{m} \cos j\delta + \sum_{n=1}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} f_n(i) \cos nj\delta \qquad (i \ge 1)$$

where S is the total applied load.

SHEAR FLOWS

Concentrated perturbation load (see fig. 3 (a)): For the shear panels in panel row j=0,

 $p_{ij} = \sum_{n=0}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} f_{\pi}(i) \sin n \left(j - \frac{1}{2} \right) \delta$

$$q_{i0} - \frac{p_{i0} - p_{i+1.0}}{2L}$$

and, for the remainder of the shear panels,

$$q_{ij} = \frac{p_{ij} - p_{i+1,j}}{I} + q_{i,j-1} \qquad (j \ge 1)$$

Distributed perturbation load (see fig. 3 (b)): For the shear panel (0,0),

$$q_{\infty} = \frac{S - 2p_{10}}{2L}$$

for the remainder of the shear panels in panel row j=0,

$$q_{i0} = \frac{p_{i0} - p_{i+1,0}}{2L}$$
 $(i \ge 1)$

and, for all other shear panels,

$$q_{ij} = \frac{p_{ij} - p_{i+1,j}}{L} + q_{i,j-1} \qquad (j \ge 1)$$

Shear perturbation load (see fig. 3 (c)): For the panel about which the load is applied,

$$q_{\infty} = \frac{3Q}{m} + \sum_{n=2}^{\frac{m}{2}} \left[\frac{f_{n}(1)}{L \sin \frac{n\delta}{2}} + \frac{2Q}{m(1+\delta_{n,\frac{m}{2}})} \right]$$

for the remainder of the shear-panels in row j=0,

$$q_{0} = \sum_{n=1}^{\frac{m}{2}} \frac{\int_{n} (i+1) - \int_{n} (i)}{2 \int_{i} \sin \frac{n\delta}{\delta}} \qquad (i \ge 1)$$

for the shear-panel (0,1),

$$q_{01} = q_{00} - \frac{2p_{11} + QL}{(L)}$$

for the remainder of the shear panels in panel row j=1,

$$q_0 = \frac{p_0 - p_{i+1,1}}{L} + q_{i0} \qquad (i \ge 1)$$

and, for all other shear panels.

$$q_{ij} = \frac{p_{ij} - p_{i+1,j}}{L} + q_{i,j-1} \qquad (j \ge 2)$$

EVALUATION OF THE TRIGONOMETRIC COEFFICIENTS $I_{A}(l)$ FOR FLEXIBLE RINGS

Basic parameters:

$$B = \frac{E}{G} \frac{t'}{l} \frac{R^2}{L^2}$$

$$C = \frac{t'R^6}{IL^3}$$

Auxiliary-parameters:

$$4+3\frac{B\delta^2}{\sin^2\frac{n\delta}{2}}$$

$$\beta_n=3+\frac{12CS}{12CS}$$

$$1 - \frac{3}{2} \frac{B\delta^2}{\sin^2 \frac{n\delta}{2}}$$

$$\gamma_n = -2 + \frac{13CS}{13CS}$$

Discriminating parameter:

$$D_n = \frac{2(\beta_n - 1)}{\gamma_n^2}$$

Trigonometric coefficients:

$$f_n(i) = \zeta_n^{-1} [\alpha_{1n} \Lambda_{1n}(i) + \alpha_{2n} \Lambda_{2n}(i)] \qquad (n \ge 2)$$

where

$$\zeta_n = -\frac{\gamma_n}{|\gamma_n|} e^{-t_n}$$

$$\Lambda_{in}(i) = \cos i\chi_n \qquad (D_n > 1)$$

$$=1 \qquad (D_n=1)$$

$$= r \sinh i \chi_n \qquad (D_n < 1)$$

$$\Lambda_{2n}(i) = \sin i\chi_n \qquad (D_n > 1)$$

$$=i$$
 $(D_n=1)$

$$=\sinh i\chi_n$$
 $(D_n<1)$

$$\chi_{n} = \frac{1}{2} \cos^{-1} \left[\frac{\beta_{n} - 1}{2} - \sqrt{\left(\frac{\beta_{n} + 1}{2}\right)^{2} - \gamma_{n}^{2}} \right] \quad (D_{n} > 1)$$

$$= \frac{1}{2} \cosh^{-1} \left[\frac{\beta_n - 1}{2} - \sqrt{\left(\frac{\beta_n + 1}{2}\right)^2 - \gamma_n^2} \right] \quad (D_n < 1)$$

$$\psi_{n} = \frac{1}{2} \cosh^{-1} \left[\frac{\beta_{n} - 1}{2} + \sqrt{\left(\frac{\beta_{n} + 1}{2} \right)^{2} - \gamma_{n}^{2}} \right]$$

Arbitrary constants for concentrated-perturbation load:

$$\alpha_{1n} = \frac{P}{m\left(1 + \delta_{n,\frac{m}{2}}\right)}$$

$$\alpha_{2n} = \frac{\theta_{1n} + 2(\gamma_n + 1)}{\hat{\theta}_{2n}} \frac{P}{m(1 + \delta_{n, \frac{m}{2}})}$$

where P is the applied load and

$$\Theta_{sn} = \zeta_n^3 \Lambda_{sn}(3) + 2\gamma_n \zeta_n^2 \Lambda_{sn}(2) + (2\beta_n - 1)\zeta_n \Lambda_{sn}(1) \qquad (s = 1, 2)$$

Arbitrary constants for distributed perturbation load:

$$\alpha_{1n} = \frac{\Gamma_{2n}}{\alpha_{1n}} \frac{\beta_n - 4\gamma_n - 2}{\Omega_{1n} \Gamma_{r2} - \Gamma_{1n} \Omega_{rn}} \frac{2S}{m(1 + \delta_{n,\frac{m}{2}})}$$

$$\alpha_{2n} = -\frac{\Omega_{1n} + \Gamma_{1n}}{\Omega_{1n} \Gamma_{2n} - \Gamma_{1n} \Omega_{2n}} \frac{2S}{m(1 + \delta_{n,\frac{m}{2}})}$$

where S is the total applied load and

$$\Omega_{sn} = \zeta_n^3 \Lambda_{sn}(3) + (2\gamma_n - 1)\zeta_n^2 \Lambda_{sn}(2) + 2(\beta_n - \gamma_n)\zeta_n \Lambda_{sn}(1) \quad (s = 1, 2)$$

$$\Gamma_{sn} = \zeta_n^{-s} \Lambda_{sn}(4) + 2\gamma_n \zeta_n^{-3} \Lambda_{sn}(3) + 2\beta_n \zeta_n^{-2} \Lambda_{sn}(\underline{2}) + (2\gamma_n - 1)\zeta_n \Lambda_{sn}(1)$$

Arbitrary constants for shear perturbation load:

$$\alpha_{1n} = -\frac{\Gamma_{2n} \frac{\beta_n - 4\gamma_n - 11}{3} \sin \frac{n\delta}{2}}{\Omega_{1n} \Gamma_{2n} - \Gamma_{1n} \Omega_{2n}} \frac{4QL}{m} \frac{1 + \delta_{n, \frac{m}{2}}}{(1 + \delta_{n, \frac{m}{2}})}$$

$$\alpha_{2n} = \frac{\Gamma_{1n} \frac{\beta_n - 4\gamma_n - 11}{3} \sin \frac{n\delta}{2}}{\Omega_{1n} \Gamma_{2n} - \Gamma_{1n} \Omega_{2n}} \frac{4QL}{m \left(1 + \delta_{\mu, \frac{m}{2}}\right)}$$

where Q is the applied load per unit length.

EVALUATION OF THE TRIGONOMETRIC COEFFICIENTS (A(I) FOR RIGID RINGS

Basic parameter:

$$B = \frac{E}{G} \frac{t'}{t} \frac{R^2}{L^2}$$

Auxiliary parameters:

$$A_n = 3B\delta^2 - 1 + \cos n\delta$$

$$B_n = 3B\delta^2 + 2(1 - \cos n\delta)$$

$$\lambda_n = \cosh^{-1} \left| \frac{B_n}{A_n} \right|$$

Trigonometric coefficients:

$$f_n(i) = \alpha_n \left(\frac{A_n}{|A_n|} e^{-\lambda_n} \right)^i$$

Arbitrary constant for concentrated perturbation load:

$$\alpha_n = \frac{P}{m\left(1 + \delta_{n, \frac{m}{2}}\right)}$$

Arbitrary constant for distributed perturbation load:

$$\alpha_n = \frac{6B\delta^2}{A_n \left(\frac{A_n}{|A_n|} e^{-\lambda_n} + 1\right)} \frac{S}{m\left(1 + \delta_{\kappa, \frac{m}{2}}\right)}$$

Arbitrary constant for shear perturbation load:

$$\alpha_n = -\frac{12B\delta^2 \sin \frac{n\delta}{2}}{A_n \left(\frac{A_n}{|A|} e^{-\lambda_n} + 1\right) \frac{QL}{m\left(1 + \delta_{n,\frac{m}{2}}\right)}}$$

APPENDIX B

BENDING MOMENT, AXIAL THRUST, AND TRANSVERSE SHEAR IN RINGS

Expressions will be developed for the bending moment, axial thrust, and transverse shear in a circular ring under tangential loads such as-those which arise from the differences in shear flow across a ring in a circular semimonocoque cylinder.

Two-cases must be considered: One case-occurs with the concentrated and distributed perturbation loads, where the ring-loading is antisymmetric about stringer j=0. The other case occurs with the shear perturbation load, where the ring loading is symmetric about panel row j=0.

CONCENTRATED AND DISTRIBUTED PERTURBATION LOADS

For the concentrated and distributed perturbation loads, the tangential loading on ring i has been written in the form of a finite trigonometric series (see eq. (16))

$$F_{ij} = q_{ij} - q_{i-1,j} = \sum_{n=1}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} b_{ni} \sin n \left(j + \frac{1}{2} \right) \delta$$
 (B1)

where

$$b_{ni} = -\frac{\Delta_{ii} f_n(i)}{2L \sin \frac{n\delta}{2}} \qquad (n \ge 2)$$

This ring load has a stepwise variation around the ring, being constant between stringers and having jump discontinuities at the stringers. The limitation that $n \ge 2$ ensures that the ring is in equilibrium.

The procedure will be to expand each term of the series (B1) in an infinite Fourier series in the variable ϕ . For each harmonic of the Fourier series, that is, for a continuous sinusoidal tangential force distribution on the ring, the moment, thrust, and shear in the ring are easily found. (See ref. 8, p. 33, for example.) On the basis of inextensional deformation and the neglect of transverse shear distortions, the results are as follows: If the tangential load on ring i is given by

$$\overline{a}_{n} \cos n\phi + \overline{b}_{n} \sin n\phi \qquad (n \ge 2)$$

then the moment, thrust, and shear-in this ring are, respectively,

$$M_{n}(i, \phi) = -\overline{a}_{ni} \frac{R^{2}}{n(n^{2}-1)} \sin n\phi + \overline{b}_{ni} \frac{R^{2}}{n(n^{2}-1)} \cos n\phi$$

$$T_{n}(i, \phi) = -\overline{a}_{ni} \frac{R}{n^{2}-1} n \sin n\phi + \overline{b}_{ni} \frac{R}{n^{2}-1} n \cos n\phi$$

$$V_{n}(i, \phi) = \overline{a}_{ni} \frac{R}{n^{2}-1} \cos n\phi + \overline{b}_{ni} \frac{R}{n^{2}-1} \sin n\phi$$
(B2)

Figure 11 shows the sign convention used in writing equations (B2).

Consider, now, one term of the series (B1). To expand this term in a Fourier series, write

$$b_{nt} \sin n \left(j + \frac{1}{2} \right) \delta = \sum_{n=0}^{\infty} (c_t)_{nt} \sin r \phi$$
 (B3)

where the $(c_{r/n})_{\sigma}$ are the Fourier coefficients. It is obvious that the first harmonic which will occur in the Fourier series in equation (B3) must be that for which r=n. The other harmonics, then, will be added to this to build up the step shape of the loading function. The convention for measuring angle ϕ in this case is illustrated in figure 12 (a). The index j can be thought of as a function of ϕ , that is; when $0 < \phi < \delta$, j=0; when $\delta < \phi < 2\delta$, j=1; and so forth.

In order to carry out the expansion, equation (B3) is multiplied through by $\sin l\phi$ and integrated from 0 to 2π

$$\sum_{j=0}^{m-1} \int_{j_j}^{(j+1)k} h_n \sin n \left(j + \frac{1}{2} \right) \delta \sin l \phi \, d\phi$$

$$= \int_0^{2\pi} \sum_{r=n}^{\infty} (c_r)_{n_1} \sin r \phi \sin l \phi \, d\phi$$

After integration, the right-hand side of this equation becomes

$$(c_t)_{nt}\pi$$

by virtue of the orthogonality of the trigonometric functions. The left-hand side becomes

$$\frac{2\sin\frac{l\delta}{2}}{l}b_{n\ell}\sum_{j=0}^{m-1}\sin n\left(j+\frac{1}{2}\right)\delta\sin l\left(j+\frac{1}{2}\right)\delta$$

on carrying out the integration. From reference 7 it can be shown that

$$\sum_{j=0}^{m-1} \sin n \left(j + \frac{1}{2} \right) \delta \sin l \left(j + \frac{1}{2} \right) \delta$$

$$= \frac{m}{2} \left[(-1)^{\frac{l-n}{m}} J_{l-n} - (-1)^{\frac{l+n}{m}} J_{l+n} \right]$$

where $J_h=1$ if h is an integer, and $J_h=0$ if h is not an integer. Thus the Fourier coefficients are given by

$$(c_{l})_{nl} = \frac{m}{\pi} b_{nl} \cdot \frac{\sin \frac{l\delta}{2}}{l} \left[(-1)^{\frac{l-n}{m}} J_{\frac{l-n}{m}} - (-1)^{\frac{l+n}{m}} J_{\frac{l+n}{m}} \right]$$

The nth term of the tangential loading on the ring is

$$b_{nl} \sin n \left(j + \frac{1}{2} \right) \delta = \frac{m}{\pi} b_{nl} \sum_{l=n}^{\infty} \left[(-1)^{\frac{l-n}{m}} J_{l-n} - (-1)^{\frac{l+n}{m}} J_{l+n} \right] \frac{1}{n} \frac{l\delta}{2} \sin l\phi \quad (B4)$$

By use of the properties of J_h this summation can be rewritten

$$b_{n} \sin n \left(j + \frac{1}{2} \right) \delta = \frac{m}{\pi} b_{n} \left[\sum_{r=0}^{\infty} (-1)^{r} \cdot \frac{\sin (rm+n) \frac{\delta}{2}}{rm+n} \sin (rm+n) \phi - \frac{\delta}{2} \cdot \frac{\sin (rm-n) \frac{\delta}{2}}{rm-n} \sin (rm-n) \phi \right]$$
(B5)

On expansion by the sum and difference formulas of trigonometry and with the use of the fact that $m\delta=2\pi$, it is found that

$$\sin (rm+n)\frac{\delta}{2} = (-1)^r \sin \frac{n\delta}{2}$$

$$\sin (rm-n)\frac{\delta}{2} = (-1)^{r+1} \sin \frac{n\delta}{2}$$
(B6)

When equations (B6) are substituted into equation (B5), the following relationship results:

$$b_{nt} \sin n \left(j + \frac{1}{2} \right) = \frac{m}{\pi} b_{nt} \sin \frac{n\delta}{2} \left[\sum_{r=0}^{\infty} \frac{\sin (rm + n)\phi}{rm + n} + \sum_{r=1}^{\infty} \frac{\sin (rm - n)\phi}{rm - n} \right]$$
$$= \frac{m}{\pi} b_{nt} \sin \frac{n\delta}{2} \sum_{r=0}^{\infty} \frac{\sin (rm + n)\phi}{rm + n}$$
(B7)

From the first of equations (B2) it is seen that if the tangential loading on the ring is given by the right-hand side of equation (B7) then the bending moment in that

ring is

$$M_n(i,\phi) = R^2 \frac{m}{\pi} b_{ni} \sin \frac{n\delta}{2} H_1(n,\phi)$$
 (B8)

where

$$II_1(n,\phi) = \sum_{r=-\infty}^{\infty} \frac{\cos{(rm+n)\phi}}{(rm+n)^2 [(rm+n)^2-1]}$$

Equation (B8) gives the bending moment in a ring which carries a tangential load distributed according to one term of the series of equation (B1). When the ring is loaded by the sum of such stepwise terms, as in equation (B1), then the moment is given by a sum of terms like (B8). The bending moment in ring i is therefore

$$M(i,\phi) = \sum_{n=1}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} R^2 \frac{m}{\pi} b_{ni} \sin \frac{n\delta}{2} II_1(n,\phi)$$
 (B9)

For completeness, the expressions for axial thrust and transverse shear can be written in a similar manner—

$$T(i,\phi) = \sum_{n=2}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} R \frac{m}{\pi} b_{ni} \sin \frac{n\delta}{2} K_1(n,\phi)$$

$$V(i,\phi) = \sum_{n=1}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} R \frac{m}{\pi} b_{\pi i} \sin \frac{n\delta}{2} L_1(n,\phi)$$

where

$$K_1(n,\phi) = \sum_{r=-\infty}^{\infty} \frac{\cos{(rm+n)\phi}}{(rm+n)^2-1}$$

$$L_1(n,\phi) = \sum_{r=-\infty}^{\infty} \frac{\sin(rm+n)\phi}{(rm+n)[(rm+n)^2-1]}$$

SHEAR PERTURBATION LOAD

In the case of the shear perturbation load, the tangential loading on ring-Lis given by the finite-trigonometric series

$$F_{ij} = q_{ij} - q_{i-1, j} = \sum_{n=1}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} a_{ni} \cos nj\delta$$
 (B10)

where

$$a_{ni} = \frac{\Delta_{ii} f_n(i)}{2L \sin \frac{n\delta}{2}} \qquad (n \ge 2)$$

Equation (B10) can be treated in a manner analogous to the handling of equation (B1). That is, each term of the series in equation (B10) can be expanded in a Fourier series. Then the moment, thrust, and shear in the ring are written immediately.

Analogous to-equation (B3), write

$$a_{nt}\cos nj\delta = \sum_{r=0}^{\infty} (c_r)_{nt}\cos r\phi$$
 (B11)

where, now, the angle ϕ is as shown in figure 12 (b). If both sides of equation (B11) are multiplied by $\cos l\phi$ and integrated from 0 to 2π , there results for the Fourier coefficients:

$$(c_l)_{nl} = \frac{2\sin\frac{l\delta}{2}}{\pi l} a_{nl} \sum_{j=0}^{m-1} \cos nj\delta \cos lj\delta$$

It can be shown (see-ref. 7) that

$$\sum_{j=0}^{m-1} \cos nj\delta \cos lj\delta = \frac{m}{2} \left(J_{\frac{l-n}{m}} + J_{\frac{l+n}{m}} \right)$$

so the:nth term of the tangential loading on the ring is

$$a_{ni}\cos njb = \frac{m}{\pi} a_{ni} \sum_{l=n}^{\infty} \left(J_{l-n} + J_{l+n} \right) \frac{\sin \frac{lb}{2}}{l} \cos l\phi \quad (B12)$$

This summation becomes

$$\widehat{X}/$$

$$a_{nt}\cos nj\delta = \frac{m}{\pi} a_{nt}\sin \frac{n\delta}{2} \sum_{r=-\infty}^{\infty} (-1)^r \frac{\cos(rm+n)\phi}{rm\oplus n}$$

which corresponds to equation (B7). Then the bending moment is

$$M(i,\phi) = -\sum_{n=2}^{\frac{m}{2}} e^{i\frac{m-1}{2}} R^2 \frac{m}{\pi} a_{ni} \sin \frac{n\delta}{2} H_2(n,\phi)$$
 (B13)

Similarly, thrust and shear are

$$T(i,\phi) = -\sum_{n=1}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} R \frac{m}{\pi} a_{nt} \sin \frac{n\delta}{2} K_2(n,\phi)$$

and

$$V(i,\phi) = \sum_{n=1}^{\frac{m}{2} \text{ or } \frac{m-1}{2}} R \cdot \frac{m}{\pi} a_{ni} \sin \frac{n\delta}{2} L_2(n,\phi)$$

where

$$H_2(n,\phi) = \sum_{r=-\infty}^{\infty} (-1)^r \frac{\sin(rm+n)\phi}{(rm+n)^2[(rm+n)^2-1]}$$

$$K_2(n,\phi) = \sum_{r=-\infty}^{\infty} (-1)^r \frac{\sin(rm+n)\phi}{(rm+n)^2 - 1}$$

$$L_2(n,\phi) = \sum_{r=-\infty}^{\infty} (-1)^r \frac{\cos(rm+n)\phi}{(rm+n)[(rm+n)^2-1]}$$

APPENDIX C

EVALUATION OF DEFINITE INTEGRALS

In order to minimize the stress energy it is necessary to investigate the following definite integrals:

$$\int_0^{2\pi} H_1(n,\phi) H_1(l,\phi) d\phi$$

$$= \int_0^{2\pi} \sum_{n=0}^{\infty} D_{rn} \cos(rm+n)\phi \quad \sum_{n=0}^{\infty} D_{s1} \cos(sm+l) \phi d\phi \quad (C1)$$

and

$$\int_{0}^{2r} II_{2}(n,\phi) II_{2}(l,\phi) d\phi$$

$$= \int_{0}^{2r} \sum_{r=-\infty}^{\infty} (-1)^{r} D_{rn} \sin(rm+n) \phi \sum_{r=-\infty}^{\infty} (-1)^{r} D_{sl} \sin(sm+l) \phi d\phi$$

where

$$D_{rn} = \frac{1}{(rm+n)^2 [(rm+n)^2-1]}$$

and where integers n and l-are limited to the following ranges:

$$2 \le n \le \frac{m}{2}$$

$$2 \le l \le \frac{m}{2}$$

Consider the relation (C1). The right-hand side can be written

$$\frac{1}{2} \sum_{r=-\infty}^{\infty} \sum_{s=-\infty}^{\infty} D_{rn} D_{sl} \left[\int_{0}^{2\pi} \cos(rm+n-sm-l)\phi \, d\phi + \int_{0}^{2\pi} \cos(rm+n+sm+l)\phi \, d\phi \right]
= \frac{1}{2} \sum_{r=-\infty}^{\infty} \sum_{s=-\infty}^{\infty} D_{rn} D_{sl} \left(\delta_{rm+n, sm+l} + \delta_{rm+n, -sm-l} \right) 2\pi$$

Now, by virtue of the limited range of the integers n and l, the following relations can be written:

$$\delta_{rm+n, \, sm+1} = \delta_{s, \, r} \underbrace{0}_{=n} = \delta_{sr} \delta_{ln}$$

$$\delta_{\ell m+n_k-sm-\ell} = \delta_{s_{\ell-\ell-\ell}-l+n} = \delta_{s_{\ell-\ell-1}-1}\delta_{l,\frac{m}{n}}\delta_{n,\frac{m}{n}}$$

Thus, when $2 \le n < \frac{m}{2}$, equation (C1) yields

$$\int_{0}^{2r} II_{1}(n,\phi) II_{1}(l,\phi) d\phi = 0 \qquad (l \neq n)$$

$$= \sum_{n=-\infty}^{\infty} D_{n}^{2} \pi = S_{n} \pi \qquad (l=n)$$

If $n = \frac{m}{2}$, the following equation is obtained:

$$\int_{0}^{2\pi} H_{1}\left(\frac{m}{2}, \phi\right) H_{1}\left(\frac{m}{2}, \phi\right) d\phi = \sum_{r=-\infty}^{\infty} \left(D_{r, \frac{m}{2}} + D_{r, \frac{m}{2}} D_{-r-1, \frac{m}{2}}\right) \pi$$

Since

$$D_{-r-1,\frac{m}{2}} = \frac{1}{\left(-rm-m+\frac{m}{2}\right)^2 \left[\left(-rm-m+\frac{m}{2}\right)^2-1\right]}$$

$$= \frac{1}{\left(-rm-\frac{m}{2}\right)^2 \left[\left(-rm-\frac{m}{2}\right)^2-1\right]}$$

$$= D_{r,\frac{m}{2}}$$

it is found that when $n = \frac{m}{6}$

$$\int_{0}^{2\pi} II_{1}\left(\frac{m}{2},\phi\right) II_{1}\left(\frac{m}{2},\phi\right) d\phi = 2 \sum_{r=-\infty}^{\infty} D_{r,\frac{m}{2}} \pi = 2S_{m}\pi$$

To summarize, then,

$$\int_0^{2\pi} II_1(n,\phi)II_1(l,\phi)d\phi = 0 \qquad (l \neq n)$$

$$= S_n\pi \left(1 + \delta_{n,\frac{m}{2}}\right) \qquad (l=n)$$

Consider the relation (C2). It is handled in a manner analogous to the treatment of (C1). Equation (C2) can be written

$$\begin{split} & \int_{0}^{2\pi} II_{2}(n,\phi)II_{2}(l,\phi)d\phi \\ &= \sum_{r=-\infty}^{\infty} \sum_{s=-\infty}^{\infty} (-1)^{r} D_{rs}(-1)^{s} D_{s1} \frac{1}{2} (\delta_{rm+n,sm+1} - \delta_{rm+n,-sm-1}) 2\pi \\ &= \sum_{r=-\infty}^{\infty} \sum_{s=-\infty}^{\infty} (-1)^{r} (-1)^{s} D_{rs} D_{sl} \left(\delta_{sr} \delta_{nl} - \delta_{s,-r-1} \delta_{n,\frac{m}{2}} \delta_{l,\frac{m}{2}} \right) \pi \end{split}$$

For $2 \le n < \frac{m}{2}$

$$\int_0^{2\pi} H_2(n,\phi) H_2(l,\phi) d\phi = 0 \qquad (l \neq n)$$

$$=\sum_{l=-\infty}^{\infty}D_{ln}^{2}\pi=S_{n}\pi \qquad (l=n)$$

If
$$n = \frac{m}{2}$$

$$\begin{split} \int_{0}^{2r} II_{2} \left(\frac{m}{2}, \phi \right) II_{2} \left(\frac{m}{2}, \phi \right) d\phi \\ &= \sum_{r=-\infty}^{\infty} \left[D_{r, \frac{m}{2}}^{2} - (-1)^{r} (-1)^{-r-1} D_{r, \frac{m}{2}} D_{-r-1, \frac{m}{2}} \right] \pi \\ &= \sum_{r=-\infty}^{\infty} \left[D_{r, \frac{m}{2}}^{2} - (-1)^{-1} D_{r, \frac{m}{2}}^{2} \right] \pi \\ &= 2 \sum_{r=-\infty}^{\infty} D_{r, \frac{m}{2}}^{2} \pi \\ &= 2S_{\underline{m}} \pi \end{split}$$

Thus the definite integral (C2) gives precisely the same result as (C1)

$$\int_{0}^{t^{2}\pi} H_{2}(n, \phi) H_{2}(l, \phi) d\phi = 0 \qquad (l \neq n)$$

$$= S_{n}\pi \left(1 + \delta_{n, \frac{m}{2}}\right) \qquad (l = n)$$

The sum

$$S_n = \sum_{r=-\infty}^{\infty} D_{rn}^2 = \sum_{r=-\infty}^{\infty} \frac{1}{(rm+n)^4 [(rm+n)^2 - 1]^2}$$

can be expressed in closed form with the aid of formula 6.495, number 2, reference 40. The result is

$$S_n = \frac{\delta^4}{12} \frac{2 + \cos n\delta}{(1 - \cos n\delta)^2} + \frac{\delta^2}{1 - \cos n\delta} - \frac{\delta^2}{4} \frac{\cos n\delta \cos \delta - 1}{(\cos n\delta - \cos \delta)^2} + \frac{5}{4} \frac{\delta \sin \delta}{\cos n\delta - \cos \delta}$$

However, the series form of S_n , because of its rapid convergence, may be more convenient than the closed form for use in computation,

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TABLE 1.-I.OAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

[B=8; C=0; m=36]

(c) Shear perturbation load about shear panel (0,0)

(b) Distributed perturbation load on stringer j=0 between rings i=0 and i=1(a) Concentrated perturbation load on stringer j=0 at ring i=0

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Shear flow, q.c.L. at station

TABLE 2.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

[H = 30; C = 6; m = 36]

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TABLE 3.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

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TABLE 4.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

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TABLE 5.-I.OAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

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1 1	38	33	1 1	38	000	-10
ı	83	6100	1.0018	0016	- 0013	90
1 1	88	88	88	2 E	1,0019	
ı	8	600	200	-,0024	- 0022	=
ı	8	: 00:	.003	1.002	0022	2
1 1	88	88	1 1	88	200	2 2
'	8	100	100.	- 813	.0012	2
,	6000	0000,	1.000	.000	9000	2
•	8	1,0003	5000.	808.	- 0003	<u>-</u>

TABLE 6.--LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

(b) Distributed perturbation load on stringer j=0 between rings i=0 and i=1B=8; C=2X10; m=36] (a) Concentrated perturbation load on stringer $j\!=\!0$ at ring $i\!\approx\!0$

(c) Shear perturbation load about shear panel (0,0)

Stringer load, p.a.f., at station	1=6	0 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1			1.5		2000 2000 2000 2000 2000 2000 2000 200
	1-5	1, ((((() () () () () () () (7=1		2.00 2.00 2.00 2.00 2.00 2.00 2.00 2.00
	ĭ	2000 2000 2000 2000 2000 2000 2000 200		, at station-	f=3		2000 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	Ę	2, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1,		Shear flow, 91, at station-	1-2		4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
		0.1.1.1 0.000 0.00		150	1		8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	ī	0.1.0.1 0.1.0.1 0.00.1			01		8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	-			_	`		0-4844664065555555
Stringer load, pii, at station-	9 1	2 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2				î	200 200 200 200 200 200 200 200 200 200
	3.5	8 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4				į	0.000 2000 2000 2000 2000 2000 2000 200
	I	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2		I mi station		2	2000. 2000.
nter load, 1	1	2		Share floor a. I. of stations.	and the same of	7.	88888888888888888888888888888888888888
E	2	2.5.5.5.5.5.5.5.5.5.5.5.5.5.5.5.5.5.5.5	-	.63		Ī	6.000 6.000
	-	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1				1	2.35 2.55 2.55 2.55 2.55 2.55 2.55 2.55
		0-00-00-00555555					0~nn+nc1-x7222222222
Stringer boal, p.s. at station	9	2000 2000 2000 2000 2000 2000 2000 200				?	2000 2000 2000 2000 2000 2000 2000 200
	1.5	200 200 200 200 200 200 200 200 200 200			-	<u>.</u>	8 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9
	ī	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1				'n	######################################
	1-3	2 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0			1, 907 6, 30	_	35255555555555555555555555555555555555
	123	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	1		Shear flow, qq.t., at	ï	89999999999999999999999999999999999999
	1	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2				ī	4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	9	3,0000000000000000000000000000000000000	•			-	344.25.25.25.25.25.25.25.25.25.25.25.25.25.
		c-04-02-100510510510	4		' 	-	0-110-20-20521122557

TABLE 7.—LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD (B=20; C=2xi0; n=36)

6	- 1			1	1 1	- 1	
9		1-6	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1			5-1	6000 6000 6000 6000 6000 6000 6000 600
ar panel	ion	f=.5	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	-		ĭ	2000 10000 1000 10
about she	uj/f., at sta	Ţ	2.000 2.0000 2.000		Shear flow, 94, at station-	<u>:</u>	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
ion losd	Stringer load, puff., at station-	F=3	1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1		ear flow, g	[=2	0. 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
erturbat	Str	1-2	4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		ŝ	1	6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
(c) Shear perturbation look about shear panel (0,0)		-	22.0 2.0 2.0 2.0 2.0 2.0 2.0 2.0 2.0 2.0			0-1	0 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
9	-				_		0-48456-2000-100
j=0		9-1	6.00 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0			3.	0.000 0.000
(b) Distributed perturbation load on stringer j=0 between rings i=0 and i=1	1	5-1	25.00 25.00				2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
nted perturbation load on between rings i= 0 and i= 1	Stringer load, p.i. at station	Ī	3 123 123 123 123 123 123 123 123 123 12		Shear flow, q.sf., at station-	ř=3	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
turbation rings i=	inger load, p	2	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		rae flow, g.s.	i=2	0.005 0.005
nited per between	Str	7.	2.00 2.00 2.00 2.00 2.00 2.00 2.00 2.00		ASS	I	\$ 4 4 5 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6
) Distrib		ī	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	-		0=1	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
e [•		0-00445510000000000000000000000000000000	<u> </u>		`	0-887450-0022822222
j=0 at		91	2006 2006 2006 2006 2006 2006 2006 2006			1:2	0.000 0.000
ıd on stringer j≔0 at	<u>.</u>	5:	2. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1.	 ¹		:	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
ord on	, at station-	ī	70222222222222222222222222222222222222		at station-	:	8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
irbation 1 ring i≃0	Stringer land, p.s. a	7	The direct of the state of the	i	Shear flow, 4.1 L. at	1	8.00 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
perturi	Stringe	1	200 200 200 200 200 200 200 200 200 200	-	Shear	-	
(a) Concentrated perturbation los		<u> </u>	2.8.8.8.8.8.8.8.8.8.8.8.8.8.8.8.8.8.8.8				######################################
Concen		0.	800000000000000000000000000000000000000			0.1	2.25 2.25 2.25 2.25 2.25 2.25 2.25 2.25
(3) (5	•	_	22222222222222	1		~,	0-484021-005255255

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TABLE 8.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

Shear perturbation load about shear panel (0,0) છ perturbation load on stringer j=0 between rings i=0 and i=1[B-1(x); C-2X10°; m-36] Distributed € Concentrated perturbation load on stringer j=0 at ring i=0

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Stringer load, p.,IL, at station-Shear flow, 94, at station Ĭ -1,2 Ī į ĩ Stringer load, p.s. at station-0.002 ĭ 3 1 ī 3 3 Stringer load, p., at station Ĭ 1.3 .

- 1	-	
	0-1	0.95 0.00 0.00 0.00 0.00 0.00 0.00 0.00
-	,	0-00-400-200222222222
	12.5	0.002 0.002 0.003
1	I	2002 2002 2002 2002 2002 2002 2002 200
Shear flow, g.d.L. at station-	[=3	4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	1=2	E GGS (1) 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
S.		1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	1	11.00 00
	_	0-8848510000000000000000000000000000000000
	ï	1400 1400 1400 1400 1400 1400 1400 1400
	1	a

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Shear flow, $q_{sl}L_{s}$ at station

TABLE 9.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

[B=300; C=2×100; m=36]

(b) Distributed perturbation load on stringer jee 0 between rings is 0 and is 1

(a) Concentrated perturbation load on stringer $j\!=\!0$ at ring $i\!=\!0$

(c) Shear perturbation load about shear panel (0,0)

Stringer load, p., I., at station-	3.	2. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1.	į	1-		
L, at station-		•			ī	5. 111111111 5. 11111111111 5. 1111111111
81	I	2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 2.00.1 3.	Shear flow, qu, at station-	1011016 10 7	1.3	8.45.000 000 000 000 000 000 000 000 000 00
r lood, p.,	1.2	41.1.1.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2		Ma during man	1-2	2.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1
Stringer	= 2	2.0 2.0 2.0 2.0 2.0 2.0 2.0 2.0 2.0 2.0	Shear fk		1	6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	ĭ	1.4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1000		0-	2
_	_					0-0444614895=555555
	9-1	6.00.5 6.00.5			13:	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
1	5.	0.123 0.658 0.658 0.018		1	1.	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
Stringer load, p.f., at station	i	0.1349 0.0550 0.		L, at station	1-3	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
nger load, p	(=3	0.238 0.518 0.618 0.618 0.618 0.608		Shear flow, quL, at station-	-2	2000 2000 2000 2000 2000 2000 2000 200
Stri	:	0.000 0.000		ŝ	1	6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6
	Ξ	0.4106 0.0023 0.0023 0.0023 0.003 0.			0:1	\$ 5000 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
	,	0-444-461-805255555			``	0-484461-005255555
	9-1	200 Maria (100 Maria (Z.	\$ 25000000000000000000000000000000000000
	f=.S	A 10 10 10 10 10 10 10 10 10 10 10 10 10			1	\$ 500 000 000 000 000 000 000 000 000 00
at station-	ī	0.000 0.000		at station-	-	M-401-000-00-00-00-00-00
Stringer load, p.s.	1	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2		Shear flow, qull, a		20-0
Stringer	3	2000 2000 2000 2000 2000 2000 2000 200		Shear fle	-	
	Ξ	\$25.55.55.55.55.55.55.55.55.55.55.55.55.5			13	480 680 680 680 680 680 680 680 680 680 6
	î	8000000000000000			į	8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
-	_	0-4445480525527555			~	0-2244000000000000000000000000000000000

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Shear flow, qull, at station-

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Shear flow, 94, at station-

TABLE 10-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

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between j=0 perturbation load on stringer rings i=0 and i=1 Distributed 3 ring Concentrated perturbation load on stringer j=0 at

Shear perturbation load about shear panel (0,0) Stringer load, pull, at atation Ĭ ĩ ĩ 9-1 į Stringer load, p.s. at stationĭ 1 1-2 Ī HUIII 401-x30=252155FX 9 111111 111111 3 load, pu, at station-Į ĩ Stringer ĩ ī 111111 ĩ

-		c-46446470577
	5.5	6 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
1	ī	8 10 2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
Sheer flow, q.d.f., at station—	į	2 0.055 0.055 0.057 0.057 0.00
er flow, q	1-2	0.000 0.000
Sp	ī	6 0438 6 1178 6
	01	0.000 0.000
	•	0-00-00-00-00-00-00-00-00-00-00-00-00-0

0-22420-205125555

TABLE 11,-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

[B = 8; C = 2 × 10; m = 35]

(0,0)		J=-E				5~2	8 (11111111 8 (1111111111111111111111111	9 9 9 9 9 9 9 9 9
(c) Shear perturbation load about shear panel (0,0)	- 00	ŝ	11111111 88888888888888888888888888888			I	8	666 666 666 666 666 666 666 666 666 66
	,(L, at static	1.1	\$1111111 \$1000 \$10		, at station-	£**)	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	000 000 000 000 000 000 000 000 000 00
	Stringer load, p., f.L. at station-	£ = 3	2 1 1 1 1 1 1 1 1 2 2 2 2 2 2 2 2 2 2 2		Shear flow, que, at station-	1-2	8 1 1 1 1 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2	11.1.1 200.1 200.1
	Strit	1.2	6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		52	ī	8 1 1 1 2 2 1 1 2 2 1 1	1111 2002 2002 2002
		I	4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1			9	8.1 12.2 12.2 12.2 12.2 12.2 12.2 12.2 1	8888
၁		`	-00400000000000000000000000000000000000			~	0=114=4461-8052572	2212
 (b) Distributed perturbation load on stringer j=0 between rings i=0 and i=1 		9-1	200 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	1000			\$685 858 858 858 858 858 858 858 858 858	1111
	Stringer load, ps, at station-	(# S		1210			\$2000 500 500 500 500 500 500 500 500 500	1111 200 200 200 200 200 200 200 200 200
		ĭ	200 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	(30)	Ob and down and the second		2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1,1,1,1 800,0 1,0 1
		- P	8.5 8.5 8.5 8.5 8.5 8.5 8.5 8.5 8.5 8.5	1,000		the mount and	2000 2000 2000 2000 2000 2000 2000 200	0.000
		<u>.</u>	8 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	300.	46		2 111111 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	2000 2000 2000 2000 2000 2000 2000 200
) Distrib		I	11111111 2255923288888888888888888888888888888888	0100.			11111111111111111111111111111111111111	
3		`	0-1187491-0082222222	2		~	0-407001-20255	22225
=0 at		91.	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	(SIO)				
(a) Concentrated perturbation load on stringer $j=0$ at ring $i=0$	1	î	848 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8			. (9 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	1 1 1 1 1 88888 88888 88888 88888
, uo pe	at station-	I				Shear How, qale, as station	6 8 8 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9	22838
ion ioad í=0		1	35.55.55.55.55.55.55.55.55.55.55.55.55.5	28		B .446. B		
orterba rin	Stringer load, p.	1	200 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	8	\	Shear 110		
ated po		=		100			1 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	1111
Joneent		0-	87000000000000000000000000000000000000	0			1111111 123 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	11111 88888 88888
(g)		₩,	C-00704040000000000000000000000000000000	4		-	0-20-001-00555	22222
	<u> </u>				-			

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TABLE 12.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

Shear perturbation load about shear panel 5=5 Ī -uol) Ĭ Stringer load, p.ill, at £ 13 12 ī છ 24222222200400-00-9 Distributed perturbation load on stringer j=0 between rings i=0 and i=13 1111111 Stringer load, p.s. at station-[P=30; C=2X10; m=36] Ĭ ij 3 0~48745600446485555 Concentrated perturbation load on stringer j=0 at ring i=0Ĵ i i i i i i i 3 1111111 Stringer load, p.1., at stationī 13 ï 7 2002000000000000000 1 ž 3

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(0,0)

ı	1	1			
		, at station	1.3	2	
		shear flow, qu,	1 = 2	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
		S	ī	2.1.1.1.1 2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.	
-			0-	8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
-			`	0-00-00-00-00-00-00-00-00-00-00-00-00-0	
-			1-3	8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
		1	1	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8	
	-	Shear flow, quL, at station-	5=3	6 11 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
			1.7	0.00 0.00 0.00 0.00 0.00 0.00 0.00 0.0	
			-	6.00 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
			0=1	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
			<u>.</u>	0-00-06-005555	
		-	ins	0	
-			711	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
		., at station-	Shear flow, qu.L. at station-	5.	8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
-		ar flow, g., I	ij		
		Sh		2000 100 100 100 100 100 100 100 100 100	
			9	E 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	

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TABLE 13.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

3 ĩ (O,O) Shear perturbation load about shear pagel 0.000 î Ī Shear flow, qu. at station-ĭ . 8 2 ž = 5 ĩ ī ĵ Ī 3 - and the same of *************** ? 9 Distributed perturbation load on stringer j=0 between rings i=0 and i=125.50 ī Stringer load, p.s. at station flow, Qu.L. at station-ĩ Ī î ĩ Shear 6.000 0.000 ï ī 3 9 11(11(11)) 3 224225500012020100 Concentrated perturbation load on stringer j=0 at ring i=091 î 1.5 Stringer load, p., at station flow, qu.L. at station-Ĭ 925220222222223 92522022222222223 9525222222222223 111111111111 3 1 î 1111111 \$ Ĩ 1111111 5 imminica 3 var-ve5255252575

TABLE 14.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

[B=30; C=2×10, m=3:]

(0,0)		91	2 1 1 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2			1-5	8 1111111111 8 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	6.88 8.88
(c) Shear perturbation load alout shear panel (0,0)	- 60	1.5	4 1 1 1 1 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5		,	ĩ	0 () () () () () () () () () (800.00
	,/L, at stati	I	2 1 1 1 1 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2		, at station	I=3	8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	888
	Stringer load, pulL, at station-	13	4.1.1.) 571.50.00 571.50.0		Shear flow, qu, at station-	i- 2	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	888
	Strik	12	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		±50	Ī	6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
		ī	2.5.00 2.00 2			0-	3 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	
(5)			-40+26-00000000000000000000000000000000000		,	·	0-46-461-462-462	2222
(b) Distributed perturbation load on stringer $j=0$ between rings $i=0$ and $i=1$	1	_	8825585525288885558	1	_	7.	255859555999999999999999999999999999999	5223
		91.0	60 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1			j	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	1111
	Stringer load, p.,, at station	5:-	21.2 2.2 2.2 2.2 2.2 2.2 2.2 2.2 2.3 2.3 2			-1	2000 000 000 000 000 000 000 000 000 00	
		I	6.50 6.50 6.50 6.50 6.50 6.50 6.50 6.50	-	at station		2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	1 1 1 1 2 2 2 8 2 2 2 8 2 2 2 8
		2	6.258 6.557 6.557 6.659		Shoar flow and, of stations.		0 0 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	32008
		2			å		1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	3888 8888 1111
Distrib		ī	4				2	7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
9	•		0-0040000000000000000000000000000000000	-		1_	0-784556666555	
<u>[</u>				_ ¦ 				
j =0 at		9	887 887 887 888 888 888 888 888 888 888	_			6 10 10 10 10 10 10 10 10 10 10 10 10 10	
on stringer j =0 at	1		255 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	- 1		. .	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	8888 1
uo pa	it station-	ij	0.13.0 0.13.0 0.23.0 0.02.0 0.02.0 0.02.0 0.02.0 0.02.0 0.03.0 0.	_			2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	8888
bation los ring i=0	ad, p., a	i=3	2			446.		ក្រ ក្រ
erturba	Stringer load, p1, at	1=2	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	1	1 0 mm	MOII JEAN	. s	
rated p		1	# 5 5 5 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9	- 1				5888 1111
(a) Concentrated perturbation load ring i=0		i=0	300000000000000000000000000000000000000				2 1111111 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	1111
3		`	0-22-22-22-22-22-22-22-22-22-22-22-22-22		,	- ·- -	0-00-00-00-00	

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Shear flow, quL, at station-

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Shear flow, 94, at station—

TABLE 15,-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

[B=1,000; C=2X10°; m=35]

Concentrated perturbation load on stringer j=0 at ring i=03

Stringer load, p.s. at station

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stringer j=0 Distributed perturbylog load on ÷

Shear perturbation logi about sigar panel (0,0) છ

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> 3. Stringer load, p. J.L. at station-Ĭ 1.3 3 ī 3~6 ĩ Stringer load, p.s. at station-Ĭ 13 ï ī 375522552500morton-0 ĩ

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		£.	Shear flow, quL, at station—	L, at station	1	
	0	2	ĩ		ī	ĭ
	0.033	0.0435	0.0320	0.0231	0.0193	0.01
	1:	0174	910	0152	0133	5
	200.	8	60	300°	5500	8
	5	250	2530	525	1500.	8
-	300	8	200	2200	8	8
•• 	6	600	28	9188		ε
· •	908	000	000		1000	8
		8	1 0013	1 812	0100.	8
*	1,0022	8		. 80.	8	2
_	180	8	1 002	.00.	1.002	8
-	1.00	8	1 00.00	100	203	8
-	83	30	1 602	100 m		8
~	200	136	188	-,0024	.0	3
-	20	1.00	.003	.00		8
=	5	1 003	6100.1	- 6018	1.83	8
<u>.</u>	9100 -	1 835	8	100,1	- 0313	8
£	1,30	8	1 800	1 000	800	8
	1,003	- 6003	1.003	. 6003	0000	8
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TABLE 16.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

9		_	400000000000000000000000000000000000000			-	8	
ar panel	-uoj	÷.	# () () () () () () () () () (,	ī	6	
 (b) Distributed perturbation load on stringer j=0 (c) Shear perturbation load about shear panel (0,0) 	Stringer load, p./L, at station-	ī	4 () () () () () () () () () (Shear flow, qu, at station-	1=3	2 111 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	
	inger load, 1	£=3	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1			1.42		
	Str	-2	8 () () () () () () () () () (ś	<u>:</u>	8 1 ()	
		ī	21 5 1 1 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2			01	11111 62525	
	~		-00-00-00-00-00-00-00-00-00-00-00-00-00			-	0-004001-622222222	
		9-1	8 10 10 10 10 10 10 10 10 10 10 10 10 10		Shear flow, qu.L., at station-	5.	11111111111111111111111111111111111111	
	1	1=5	26.28.88.88.88.88.88.88.88.88.88.88.88.88.			ī	20 11 11 11 11 11 11 11 11 11 11 11 11 11	
	Stringer load, p.s. at station-	Ĩ	88399889998999999999999999999999999999			÷	8 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
	nger load, p	<u>.</u>	200 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5			1.2	6 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8	
	Strdi	1	6.559 6.559			Ξ	0.05 (1) (1) (1) (2) (2) (2) (2) (2) (2) (2) (2) (2) (2	
		ī	8.3.2.2.3.3.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1			01	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
Ē	-	•	0-46-46-465-365-355-3			~ 	0-00-00-00-00-00-00-00-00-00-00-00-00-0	
j =0		9-1	200 200 200 200 200 200 200 200 200 200				1-3	11111111 1950 1950 1950 1950 1950 1950 1
(a) Concentrated perturbation load on stringer $j = 0$ at ring $i = 0$	1	1.5	99 9 8 8 8 8 9 9 9 8 8 8 8 8 8 8 8 8 8	!		1	0 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
no puo	at station	ĭ	20 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2		quL, at station-	8.1	2	
ition los ; ir±0	ed, p.,	1.3	200 200 200 200 200 200 200 200 200 200				1	
erturbat at ring	Stringer load, p.,, at station-	1	0.00 0.00 0.00 0.00 0.00 0.00 0.00 0.0		Shear flow,	1	2000 2000 2000 2000 2000 2000 2000 200	
ntrated 1		ī			₽ \$	1	9 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3	
Concer		12	800000000000000000000000000000000000000	•		011	\$2255555555555555555555555555555555555	
(3)		~	0-48-40-80522222222	2		· ·	0~20700100000000000000000000000000000000	

TABLE 17.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

(b) Distributed perturbation load on stringer j=0 $(B = 30; C = 2 \times 100; m = 36)$ (a) Concentrated perturbation load on stringer j=0 at

			(
(0,0)		9	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1			13	2
about shear pane	1=5	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	***************************************	I	I	0.000 0.000	
about she	(c) Shear perturbation load about shear panel Stringer load, p.ul., at station—	I	1 1 1 1 (r, at station	<u>.</u>	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
ion load		5.1	0.000 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		Sbear flow, qu, at station-	12	6 00 00 00 00 00 00 00 00 00 00 00 00 00
erturbat		(m2	4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	•		ī	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
Shear 1		ī	0.000 0.000			0-1	2000 2000 2000 2000 2000 2000 2000 200
ခ်						`	0-4844864890-6855564
,		9.	2000 2000 2000 2000 2000 2000 2000 200			3.	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
- -	con rings i=0 and i=1 Stringer load, p., at station-	Ž.	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		Shear flow, q., L, at station-	ī	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
		ĭ	2.00.00 2.00.0			[-3	0.000 0.000
between rings i=0 and	inger load, j	<u>:</u>	9.00 1.00 1.00 1.00 1.00 1.00 1.00 1.00			1-2	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
between	Str	£ 1	20.00000000000000000000000000000000000			1	0.000 0.000
		ī	6 25 25 25 25 25 25 25 25 25 25 25 25 25			0-1	2 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
	``		0-00-00-00-00-00-00-00-00-00-00-00-00-0			_	0-004-060-0055885555
		9-1	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0			i=5	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
,		í-5	0.00.00 0.00.00 0.00.00 0.00.00 0.00.00 0.00.0			ī	00000000000000000000000000000000000000
	station-	ī	0.000 0.000		station	E 3	75 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
ring i=0	ad, p.i. a	5.	0.00% 0.00%			-	
ring	Stringer load, p., a	:	0.00 0.00 0.00 0.00 0.00 0.00 0.00 0.0	·	Shear flow, q., L. a		2000 2000 2000 2000 2000 2000 2000 200
·		1	2,22,22,22,22,22,22,22,22,22,22,22,22,2	-		=	B.738
		0-1	800000000000000000000000000000000000000			0-1	1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1
	7		0-110-100-100-100-100-100-100-100-100-1	-	-	ا۔ 	0-00-00-000-000-00-00-00-00-00-00-00-00

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Shear flow, quL, at station

qu, at station-...

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(0,0)

TABLE 18.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

100; C = 2 × 104; m = 36]

Distributed porturbation load on stringer j=0 between rings i=0 and i=13

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Shear perturbation load about shear panel î Stringer load, pull, at station-Ĭ ĩ 12 ĩ ত্ 94 Stringer load, ps, at stationī £#3 -5 Ī C-107701200012012012 91 Concentrated perturbation load on atringer j=0ring i=0ï ĭ BUIG Stringer load, p.,. , ř 888888888 1111111 3 01

Shear flow,	1=2	0 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
Z.	ī	2 1 1 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
	9	100 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	`	0-66-46-405255555
	· -	
i	25	0 0 0 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	1	0.000 0.000
Shear flow, qull, at station-	1	6.00 1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.
car flow, qu	1 22	6. 6855 675 675 675 675 675 675 675 675 675 6
- 5	1	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0

4251873568468+8F+0

0~110402102222222222

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TABLE 19.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAL

Shear perturbation load about shear panel (0,0) છ Distributed perturbation load on stringer j=0 between rings i=0 and i=1[B=300; C=2X10°; m=36] 3 (a) Concentrated perturbation load on stringer j=0 at ring i=0

	9-1	20.00 20.00		fw5
	f=-5	24.00 - 1 - 0 - 0 - 0 - 0 - 0 - 0 - 0 - 0 -	•	ij
/L, at statio	Ξ	6.00% 6.00%	at station-	1=3
Stringer load, p.,fL, at station—	1-3	2-1-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1-1 2-1 2	Shear flow, qu, at station	/m2
Strin	-2	4 () () () () () () () () () (£	1
	Ī	2.5.5.6.1 2.5.6.6.1 2.5.6.6.1 2.5.6.6.1 2.5.6.6.1 2.5.6.6.1 2.5.6.1 2.		0-)
*				`
	9-1	45 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4		
Stringer load, ps., at station—	i=5	1.1237 1.2237 1.		-
	1-1	23.5 23.5 23.5 23.5 23.5 23.5 23.5 23.5	Shear flow. au.L. at station-	
nger load, p	E+!	2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.	ar flow. au.l	
Stri	1	dS.		
	[]	0.000 0.000	-	,
	-	0-0040000000000000000000000000000000000		`
	9-	20047		1
	ins.	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		-
4 station—	Ĭ	2.000 000 000 000 000 000 000 000 000 00	station-	
ed, p.i. a	f=3	25.00 25.00	go L. at	
Stringer load, p.1, at	2	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	Shear flow, och, at	
i	1	8. 3. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1.	2	-
	5	800000000000000000000000000000000000000		
	<u> </u>	0-00-00-00-00-00-00-00-00-00-00-00-00-0	-	-
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	1	2		5000	Š	648	58	.0462	5	2010.	, 000.	. 872	1,0103	1,010	1,089	100	1 8 1 1	1, (6)21	1,000	0100	¥0.	-
•	`			o =		~	•	•••	9		*	٥	2	=	2	=	=	2	2	12	2	
		?		0.0105	9	¥10,	6	3F85,	200,	300	0100	-,0021	1,002	-,0003	.003	- 003	1.003	1,002	02(m)*i	1.0013	#8'-	-
1	-	ī		0.015	9210	0610.	8	58		2000	012	1,0024	-, 0.032	389,	268	.000	1,000	.002	0021	0313	1010	-
7. at station	/, at station			0,0211	9120	¥,	300.	1500	2200.	000	9100	1000	-,039	980	1,842	98.	900	0000	1,0023	1.89.1	1 0005	
Shear flow, qu.l., at station-				0000	ē.	98.5	. D.92	6100	\$100,	99.1	.002	0032	0400	100.	3.0	- 8042	. 0033	.003	1,0024	-,0015	.00°.	
ďS		Ī		0.0643	0000	.0162	933	96,	2100.	9000	2007	.003	.,0043	1.00	3,63		689	-,003	1,0023	1.00.	300,1	
		ę.		0000	3180	910.	Ž.	:go	300	-,025	900	960	358	3	5400.1	1,0043	1,003	1000	:003	100	1 0005	
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_	7		-	9			ء.	3	23		×	_	Ŀ	ຍ			2	20	2			
1				é	20.0	010,	S,	3	8	90,	000	3	100	1.00	1,000	. ogg	98.1	5	į.	8,1	ĕ.	
		1		0.0127	.0157	1210	200	933	88	2000	8,1	1,000	0000	-,003	1 000	, 003 1	-,0032	- 002	000	S13	, 900	

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TABLE 20.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

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	9-	2 1 1 1 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2			ž.	Q (1,1,1,1,1,1,1,1,1,1,1,1,1,1,1,1,1,1,1,	
Stringer load, PulL, at station—	1.	41.1.1 81.9.1 81.9 81.9			ī	A 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
	ī	21.1.1 21.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.		at station-	F.	20000000000000000000000000000000000000	
	F.3	4.1.1. 4.6.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.		Slwar flow, 44, at station-		2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
	<i>i</i> -2	411 266 266 266 266 266 266 266 266 266 2				ī	10.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.
	ī	\$1. \$1. \$1. \$1. \$1. \$1. \$1. \$1. \$1. \$1.			î	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	
	`	-4446-495555555	-	! -		e-un-ver-xe5=555555	
	9=1	2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1			1.5	2 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
		6 2 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	_		1	Decaration	
, at station	ĭ	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	i	Shear flow, q.,I., at station-	1.3	-072-1-0000xmv2+0x+7m	
Stringer load, p.,, at station-	 	25 25 25 25 25 25 25 25 25 25 25 25 25 2			1-2	37701-3770-01-073-37	
Strin	-2	######################################	'		Ī	0700cx07~cxx07200	
	ī		-		0.1	2000 2000 2000 2000 2000 2000 2000 200	
		0-0040404055555555555555555555555555555	- '	_	-	c-48-25-15-15-15-15-15-15-15-15-15-15-15-15-15	
	9 .	123 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	-		3.	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
	6-1	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1			ī	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
t station	ĭ	A 227 C C C C C C C C C C C C C C C C C C		Shear flow, 4.,L, at station-	.: .:	00000000000000000000000000000000000000	
od, p., 5	i.3	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2		'. 4., L. at		232225255552999995=52	
Stringer load, pu, at	i=2	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2		hear flow	1	2000 2000 2000 2000 2000 2000 2000 200	
	Ţ.	\$2555555555555555555555555555555555555	-	-	=	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
	0 - !	80000000000000000000000000000000000000			01	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
		0-00-00-00-202222222			`	0-0740000000000000000000000000000000000	

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Shear flow, qull, at station

Shear flow, 44, at station-

TABLE 21.-LOAD, DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

(c) Shear perturbation load about shear panel (0,0)

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o.d.L. at station

j=0			0.0332
stringer = 1	<u>.</u>	Şi	0.0575
ı load on ı0 and i≖	Stringer l'xid, p.s. at station-	I	0.0639
turbation rings i=	nger load, p	[=2 [=3	0 034
outed per between	, lu	-2	9100 0
 (b) Distributed perturbation load on stringer j=0 between rings i=0 and i=1 		ī	0,2241 0,0016 0,0754
=		•	0-
no at		ĩ	0.0315
5			
ringe		ž.	0.0551
ıd on stringe	t station-	1-1	0.0003 0.0551
tion load on stringe g i=0	œd, pu, at station-	in3 in4 in5	0.0650 0.0633 0.0531
xerturbation load on stringe ring i=0	Stringer load, p.4, at station-	i=2 i=3 i=4 i=5	0.0556 0.0550 0.0500 0.0551
trated perturbation load on stringe ring i=0	Stringer load, p4, at station—	in 2 1 2 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	0.00+H 0.0856 0.0659 0.0000 0.0051
Concentrated perturbation load on stringer j = 0 at ring i = 0	Stringer load, pu, at station-	1-0 1-1 1-2 1-3 1-4 1-5 1-6	0.5000 0.0414 0.0556 0.0559 0.0003 0.0051 0.0051 0.0050 0.0050 0.0051 0.0051 0.0050 0.

E

Stringer load, p.	1.3	0.000 0.000
	1-2	0.00 0.00 0.00 0.00 0.00 0.00 0.00 0.0
	ī	2010 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000
	` '	
	9-1	28.00 28.00
į	Ş <u>-</u>	0.05.5 0.05.5 0.05.5 0.01.5 0.
atringer load, p.s. at station-	I	0.053 0.053
nger load, p	f=3	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
NIA.	1-2	0 00 00 00 00 00 00 00 00 00 00 00 00 0
	Ē	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
~		0-48446-205255555
		AN-20/40-4000-00
æd, p.,, at station	9-!	2.00 2.00 2.00 2.00 2.00 2.00 2.00 2.00
	}=\$	0.000 0.000
	1.1	0.000 0.000
00d, p4, 8	£.	0.000 0.000

	!_	
	s	2 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
1	Ξ	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
L, at station	f=3	2 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
Sbear flow, 94L, at station—	i=2	2 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
ą,	ī	2 111111111111111111111111111111111111
	0~)	2. 1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1
	`	0-48744
		22222122312232222

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TABLE 22,-LOAD DISTRIBUTION DUE TO A CNIT PERTURBATION LOAD

[B=30; C=2X10³; n=36]

(0,0)		9=1	2000 2000 2000 2000 2000 2000 2000 200			j=5	2000 2000 2000 2000 2000 2000 2000 200
(e) Shear perturbation load aboutshear panel (0,0)	- lou-	1.5	10 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		ı	1-1	0.00 0.00 0.00 0.00 0.00 0.00 0.00 0.0
	p.,//, at stal	ĭ	2.000.000.000.000.000.000.000.000.000.0		Shear flow, qu, at station-	£=1	0.00-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1
	Stringer load, p.,/L, at station-	[=3	2111 612 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5		hear flow, q	f=3	0.020 0.020
	ĕ	Ĭ=2	411, (1) (1) (1) (1) (1) (1) (1) (1) (1) (1)		S.	1-1	1.21.0 1.25.0 1.
c) Shear		ī	4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1			<i>9−1</i>	0.0572 0.0727
٤		<u> </u>	-44-45-0052552555				0-004-00-x00=001145FF
j=0		91	8.85.85.85.85.85.85.85.85.85.85.85.85.85				2.00 2.00 2.00 2.00 2.00 2.00 2.00 2.00
(b) Distributed perturbation load on stringer j=0 between rings i=0 and i=1	-6	5-1	0.0027 0.0037 0.	1	1	I	0.016 0.016
ted perturbation load on between rings i=0 and i=1	Stringer lawl, pu, at station	I	0.0570 0.0570 0.0570 0.0570 0.01570 0.		Shear flow, q.d.L, at station-	£.	2 (1984) 1 (198
turbation ring<	Inger lowl,	7	5.55.55.55.55.55.55.55.55.55.55.55.55.5			12	100 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
between	Sir	1-2	0.000 () () () () () () () () ()	1		Ī	200 200 200 200 200 200 200 200 200 200
) Distril	_	Ī.	0.000 (1.			9	\$200.1111115 \$200.1111115 \$200.111115 \$200.111115 \$200.1115 \$200.1115
= ;	-		0-9846612055555	1	·	`	0-484461-805=555
j=0 at		9=	0.000 0.000	-		Şesj	200 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
stringer	<u>.</u>	.1	0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.00000 0.00000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.000000		1	1	8 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
oad on	, at station	Ī	2.00 0.00 0.00 0.00 0.00 0.00 0.00 0.00		at station-	1=3	\$200 \$200 \$200 \$200 \$200 \$200 \$200 \$200
(a) Concentrated perturbation load on stringer j=0 at ring i=0	Stringer laul, p.j. at station		88288882====885==555 	1	Sheur flow, qul.,	12.2	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	String	-	201 0 201 0		Shear		######################################
			200 000 000 000 000 000 000 000 000 000				
Come		0.1			! !	9	4
3		_]	0-110-40-1-405-1211111111111111111111111111111111111)		٠.	C-00440440052552555

TABLE 23,-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

(c) Shear perturbation load about shear panel (0,0)

[B=100; C~2X10; m=36]

(b) Distributed perturbation load on stringer j=0 between rings i=0 and i=1

(a) Concentrated perturbation load on stringer j=0 at ring i=0

-						
Sirinker load, pull, at station-	9=1	4. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1.		15		4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	f=S	0.00 0.10 0.10 0.00 0.00 0.00 0.00 0.00	,	:		6 10 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	ĭ	6110 4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	, at station	1	-	6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	<u>.</u>	6.000 (1.000	Shear flow, 9,,, at station-	[-2	-	5.1.1.1 5.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.
	(=2	4.11.4 (000) (000) (000) (000) (000) (000) (000) (000) (000)	55 	1	-	27 - 0 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1
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			1	-		
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o,f. at statle	I	0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0	2000	io, at state	Ĭ.	0.000000000000000000000000000000000000
Stringer load, p.s. at station—	Ĩ.	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		PIRTSE HOW, Q.J.D. at Station	ï	\$ 1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.
Strl	i - 2	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	45	XX.	ī	5. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1.
	ī	24.00 26.00 26.00	-	-	9	45455989955995988 2
	`			``		0-007-04-002202255
	9-1	\$25.50 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0			2	200 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
	3	2.00 2.00 2.00 2.00 2.00 2.00 2.00 2.00	-	-	ĭ	2000 2000 2000 2000 2000 2000 2000 200
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, p., at	ĩ	2.00 2.00 2.00 2.00 2.00 2.00 2.00 2.00		2 Kr. 31 y	7	4 11111111111
Stringer load, p.,, at	~	84.00 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	,	Shear How, quit, at station		2.00 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
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	e.	800000000000000000000000000000000000000			9	0.52 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
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TABLE 21,-LOAD DISTRIBUTION DUE TO A UNIT/GERTURBATION LGAD

[B=300; C=2 X 104; M=36]

9= 3.5 90 perturbation lead about shear panel Ĭ Stringer load, pull, at station-Shear flow, qu. at Matlon-7 Ī 7 17 17 ī Shear Ξ 1 છ 91 Distributed perturbation load on stringer j=0 between rings i=0 and i=1amon p.i. at station-Shear flow, qull, at station-Ī Stringer load Ĩ Ī Ī THURST -8844864005=384856 X-1922-201-00x-190-00 on stringer j==0 at 9= 0.135 0.025 5 THEFT 11111111 perturbation load Stringer load, F.,, at ï MANGE ï Concentrated 1///1111 9 3 0-00-00-00-0022022022

TABLE 25.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

[B. LOW; C-2X109; m-36]

Shear perturbation load about shear panel (0,0) Stringer load, pigli, at station-Ĭ

Distributed perturbation load or stringer j=0 between rings i=0 and i=1

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Concentrated perturbation load on stringer j=0 at ring t=0

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Stringer load, p.,, at station-7 ĩ 3 0-224401-6955 91 į Stringer foul, p.j. at station-I 1111111 Î ĩ 288888 111111 ĩ 2

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22222	
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Shear flow, qull, at station—	ī	6.00 (1.00 (
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	ī	0.000						
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0-00749008002222222

Shear flow, quL, at station

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	, at station	f=3	
	Shear flow, qu, at station-	í-2	-
_	₹	ï	
		i=0	
_			

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-	0-44-461-49028823553

TABLE 26-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

(H = 8; C = 2 × 10* 16 + 36)

91 (0,0) panel ï ĭ perturbation load about shear p.dl., at station Shear flow, g.j. at station ĩ Į. Stringer load, 5 1-2 ï ĩ Shear ē 3 9 stringer 8.<u>r</u> , qull, at station 1 Distributed perturbation between rings ix ĩ Shear 111111111 Ī 11 3000000000 THE COUNTY OF THE 3 stringer j=0 10771111116 -rotters te. Ling . con red ē Ī o pad o 1111111111 3 1 perturbution l ring i == 0 111111111111 £££££\$\$\$\$\$\$\$\$\$\$\$\$ Stringer î Concentrated . 72962888883838388888 1 1111111111111111 E

TABLE 27.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

113-30, C-2X10; 7-401

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	9=1	4 0 0008 1 1 1 0 0008 1 1 0 0009 1 0 0009 1 0 0009 1 0 0009											
į	1-5	4 1 1 1											
/L. at static	I	1000 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	at station-										
er load, p.	er load, p.,	er load, p.,	E=1	0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000	Shear flow, q.,, at station-								
Strin	[-3	6.000 6.000	£										
	ĩ	88.00 8.00 8.00 8.00 8.00 8.00 8.00 8.0											
-	•	-464464466464664664											
	9-,	0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.00000											
Stringer load, p.s. at station-	on, at station-		į	(=-S	0.0527 0.0526 0.	1							
		ij	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	at station									
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	í=2	20115. 10035. 10	She										
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load, p.r.	load, p.r.	ï		Shear flow, q.,L. at									
Stringer	1-2		Shear flo										
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	`	0-00-0000000000000000000000000000000000	1										
	Stringer load, p.n. at station— Stringer load, p.n. at station— Stringer toad, p.n.f., at station—	Stringer load, p., at station— 1=1 i=2 i=6 i=1 i=3 i=4 i=5											

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į.	I	20000000000000000000000000000000000000
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Shear flow, qull., at station-	í = 2	0.0216 0.0216 0.0216 0.0216 0.0228 0.0288 0.0288 0.0288 0.0288 0.0288 0.
4S	1	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
	0	0.1000 0.0000 0.00
	`	0-4044444205=555
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10		

0-00-00-00-00-00-00-0

•		σ,	Shear flow, q.,, at station—	, at station		
`	0-1	[4]	2-1	i=3	11	1-5
-	0.7274	0.0994	0 0211	0.0078	0.0031	0 0017
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=	000	8	8	6000	8	900
2:	88	888	88	98	8	8
21	38	38	38	88	88	38
•	38	38	38	38	38	33
•	88.	3	RING	3	3	mon.

TABLE 28.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

(a) Concentrated perturbation load on stringer j=0 at

(b) Distributed perturbation load on stringer j=0 between rings i=0 and i=1

	-			~~~·		
(0,0)	Stringer load, p.,/L, at station—	91	2. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1.	at station	5-1	6. 11.1.1 2. 12. 12. 12. 12. 12. 12. 12. 12. 12. 1
Shear perturbation load about shear panel (0,0)		î=S	010 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0		ī	0 (11) (10) (11) (10) (10) (10) (10) (10
		ĭ	4.1. 2.0000 2.0000 2.0		(13	6 1 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
		f=3	2. 1. 1. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2.	Shear flow, qu, at station	in 2	2000 2000 2000 2000 2000 2000 2000 200
		}=5	11.0 10.0 10.0 10.0 10.0 10.0 10.0 10.0	S	Ī	2 1 1 1 1 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2
		ī.	4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		i=0	2 ± 2 ± 2 ± 1 ± 1 ± 2 ± 2 ± 2 ± 2 ± 2 ±
<u> </u>	~]		-48466-8955555		<u> </u>	
0=0	Stringer load, p.s. at station—	9.	8.25.25.25.25.25.25.25.25.25.25.25.25.25.		1	0.000 0.00000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.00000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.00000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.00000 0.0000
stringer = 1		?	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2		77	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
 (b) Distributed perturbation load on stringer j=0 between rings i=0 and i=1 		ī	20 00 00 00 00 00 00 00 00 00 00 00 00 0	Sheer flow m.f. at station		### ##################################
			0.1433 0.001 0.001 0.001 0.001 0.001 0.001 0.000 0.0001 0.000001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.00001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001 0.0001		an inches	0.00 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
		~	25 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		1	2 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
		ī	8 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9		9	6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
		_	0-48446-405555555		`	0-00-001/005505555
Concentrated perturbation load on stringer $j=0$ at ring $i=0$	Stringer load, p.,, at stution-	ĩ	2.00 (1.00 (6 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.
		1=5	8.25.25.25.25.25.25.25.25.25.25.25.25.25.			5 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
		ĭ	4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1			
			25.50 25.50		4.4	. 86222566565552222
		?	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		Silvar tion, quite in	
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	<u></u>					

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TABLE 29.-LOAD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

perturbation load about shear panel (0,0) ĩ Ĩ Shear flow, qu, at station 0.500 0.000 ĭ 1.3 . 122 23 Ī Shear 0.35.0 1.02.0 1.02.0 1.02.0 1.00.0 1. 0.453 11.00 Ĩ 9 9=1 î 6.136 6.256 ĩ į 11111111111 g.!! flow, qu.L, at station Distributed perturbation load of between rings i=0 and 0.021 0.025 Į ĩ indidition i 1.3 3 CARD CONTRACTOR CONTRA 0,4132 (600) ī į Ê $\overline{}$ ĩ មិវិជីជីវិជីជ loul, p.r. at statin ĩ Ĩ A CONTRACTOR OF THE PROPERTY O ĩ finifith Ī ī ĩ ε 0748-001-005-25255

U S. GOVERNMENT PRINTING OFFICE: 1958



TABLE 30,-1,0AD DISTRIBUTION DUE TO A UNIT PERTURBATION LOAD

[B=1,000; C=2 × 10⁶; m=36]

			The second secon				
(0,0)	Stringer load, p., f., at station-	9-1	4.0 0575 1.0 0105 1.0 0105 1.0 0105 1.0 0105 1.0 0105 1.0 0005 1.0			<u>:</u>	6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
(c) Shear perturbation load about shear panel (0,0)		. <u>.</u>	2.1.0.1 (2.0.0) (1.0.0		,	I	0.004 0.004
		I	2. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1.		at station-	f=3	A 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
		1	A 22.25. 10. 10.25. 10.25. 10.25. 10.25. 10.25. 10.25. 10.25. 10.25.		Sbear flow, q.,, at Matton—	i=2	A 1 1 1 5 2 5 2 5 2 5 5 5 5 5 5 5 5 5 5 5
		1-2	6.015 10.01 10		£	1	2.10 2.10 2.10 2.10 2.10 2.10 2.10 2.10
		ī	4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4			0 - i	2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.
<u> </u>					*****	`	0-4444000000000000000000000000000000000
	Stringer load, p.i. at station-	٠	226522222222			Ţ.,	226728883728727
(b) Distributed perturbation load on stringer j=0 between rings i=0 and i=1		3-1	25.00 25.00	_	Shear flow a. J. at station	6.01	2000 2000 2000 2000 2000 2000 2000 200
		į	8	1		1	2000 2000 2000 2000 2000 2000 2000 200
		Ī	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2			į	A 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
		5.3	# # # # # # # # # # # # # # # # # # #			î	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
		1-2	## ## ## ## ## ## ## ## ## ## ## ## ##		. £	1	4. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2.
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د			45582510005100=00=000	,			
(a) Concentrated perturbation load on stringer $j=0$ atring $i=0$	Stringer load, p., at station-	91	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	-		1	100 100 100 100 100 100 100 100 100 100
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